

Design in State Space (time domain)

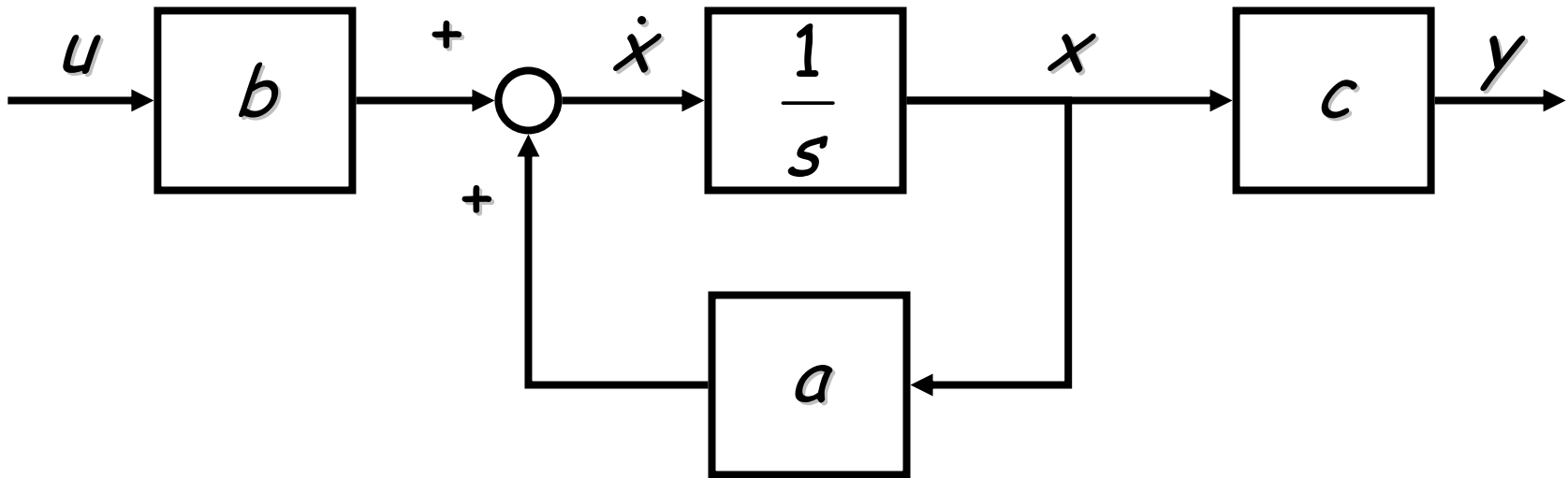
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- State space description
- state feedback
- pole placement
- optimisation

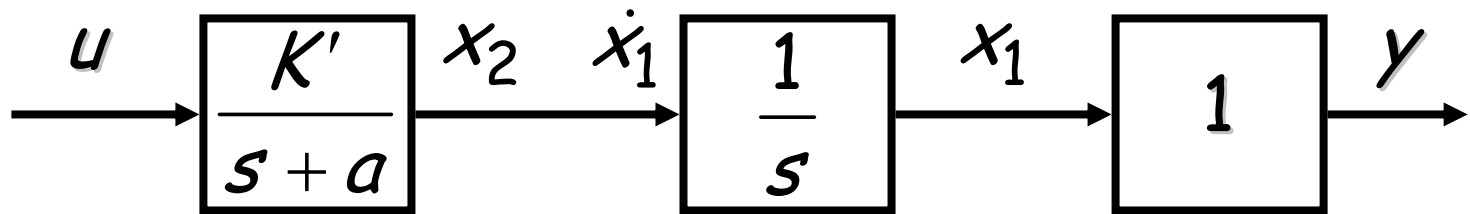
First-order system



$$\dot{x} = ax + bu$$

$$y = cx$$

Second order system



$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = -ax_2 + K'u$$

$$y = 1x_1$$

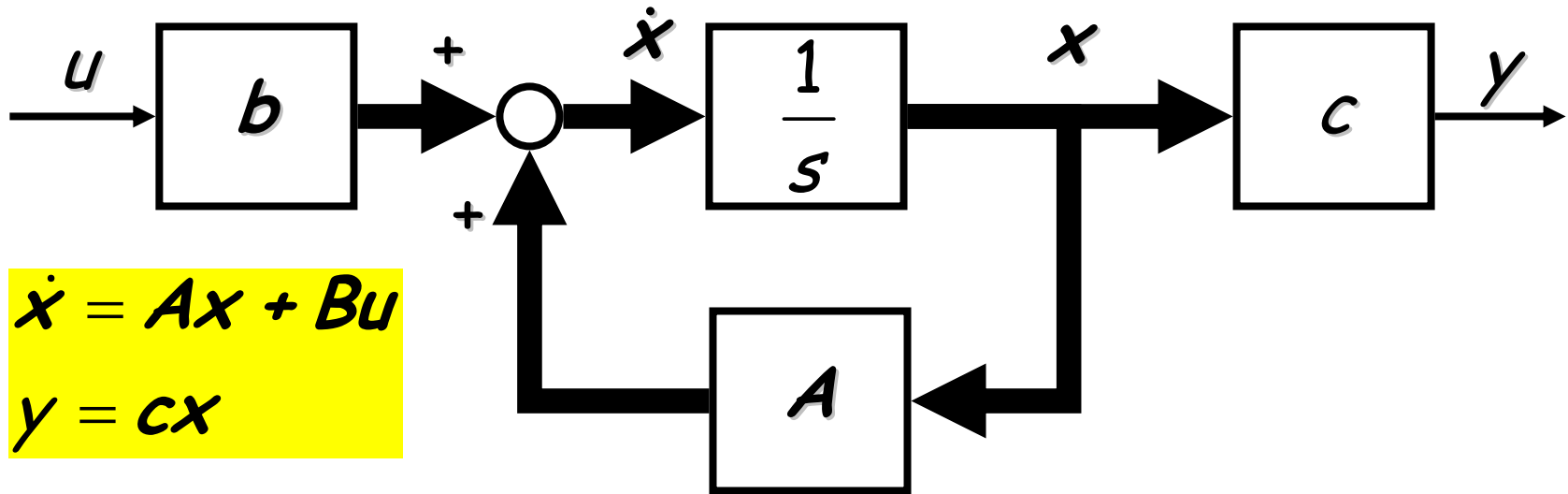
$$\dot{\mathbf{x}} = \begin{pmatrix} 0 & 1 \\ 0 & -a \end{pmatrix} \mathbf{x} + \begin{pmatrix} 0 \\ K' \end{pmatrix} u$$

$$y = (1 \ 0) \mathbf{x}$$

$$\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{b}u$$
$$y = \mathbf{c}\mathbf{x}$$

State-space description

Set of first-order systems



For **SISO** systems:

u = input signal (scalar)

y = output signal (scalar)

x = state vector ($n \times 1$)

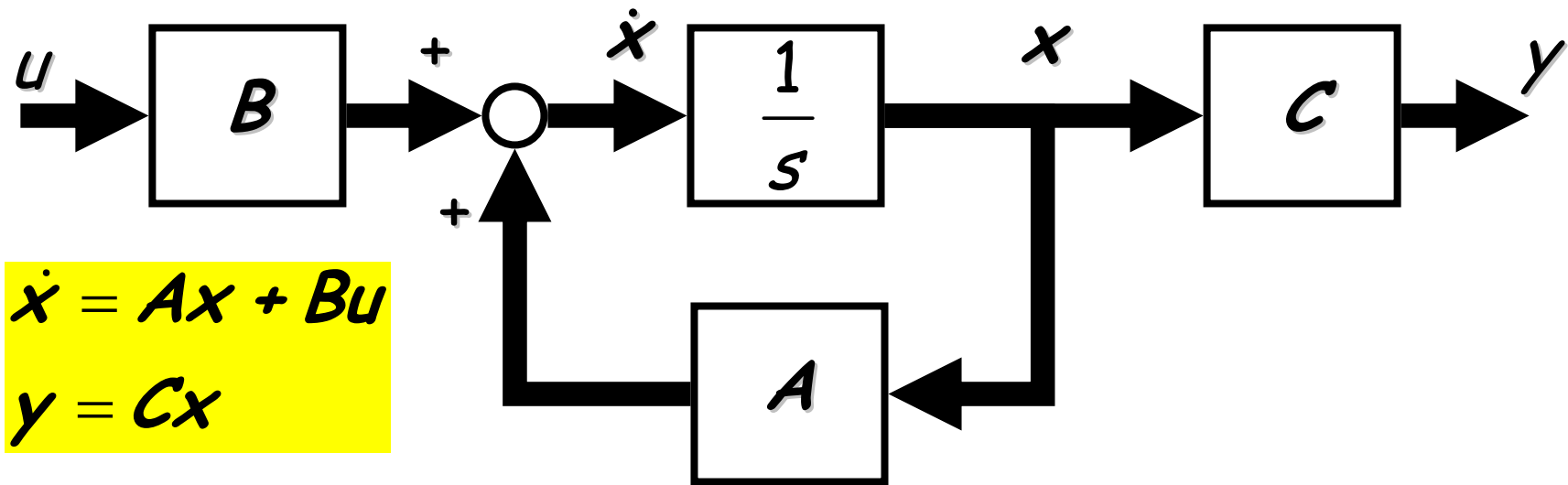
For **SISO** systems:

A = system matrix ($n \times n$)

b = input matrix ($n \times 1$)

c = output matrix ($1 \times n$)

Set of first-order systems



For **MIMO** systems:

u = input vector ($m \times 1$)

y = output signal ($p \times 1$)

x = state vector ($n \times 1$)

For **MIMO** systems:

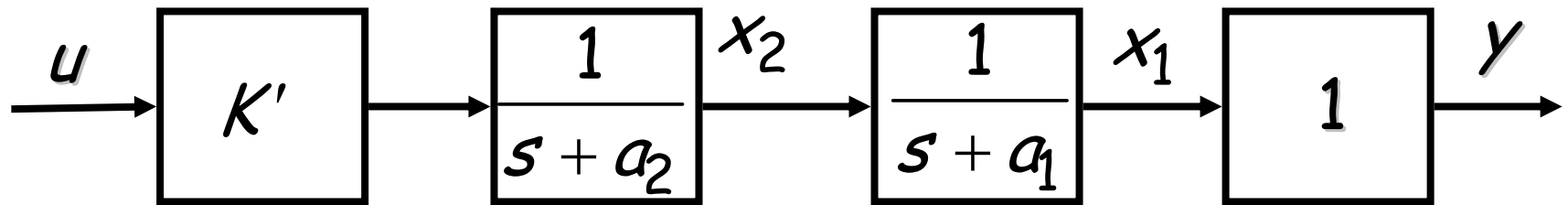
A = system matrix ($n \times n$)

B = input matrix ($n \times m$)

C = output matrix ($p \times n$)

- The state ($x(t_0)$) of a system at $t = t_0$ is the minimal amount of information that is necessary to describe the behaviour of the system for $t > t_0$, if also the input(s) and the state equations are known

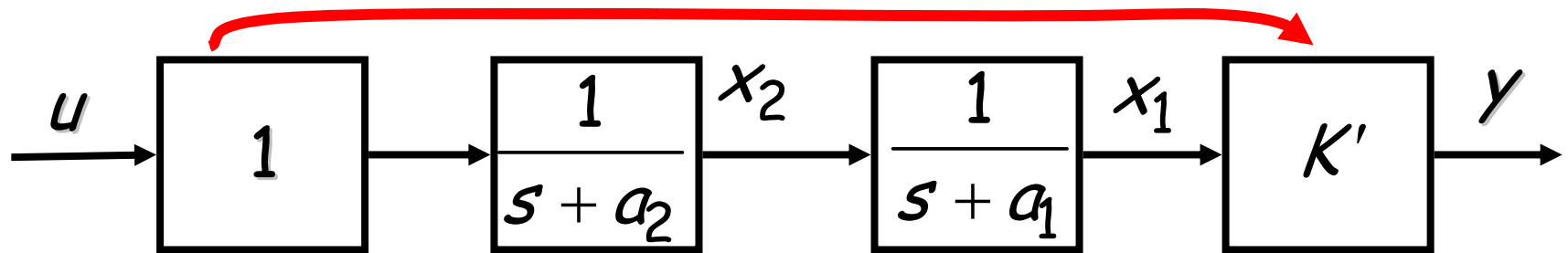
- State variables are not unique
 - any linear combination of state variables is a state variable again
- E.g. the initial conditions of the integrators in the system



$$\begin{aligned} \dot{x}_1 &= -a_1 x_1 + x_2 \\ \dot{x}_2 &= -a_2 x_2 + K'u \\ y &= 1x_1 \end{aligned} \quad \rightarrow \quad \begin{aligned} \dot{x} &= \begin{pmatrix} -a_1 & 1 \\ 0 & -a_2 \end{pmatrix} x + \begin{pmatrix} 0 \\ K' \end{pmatrix} u \\ y &= (1 \ 0) x \end{aligned}$$

eigenvalues at the diagonal

Series form (alternative)



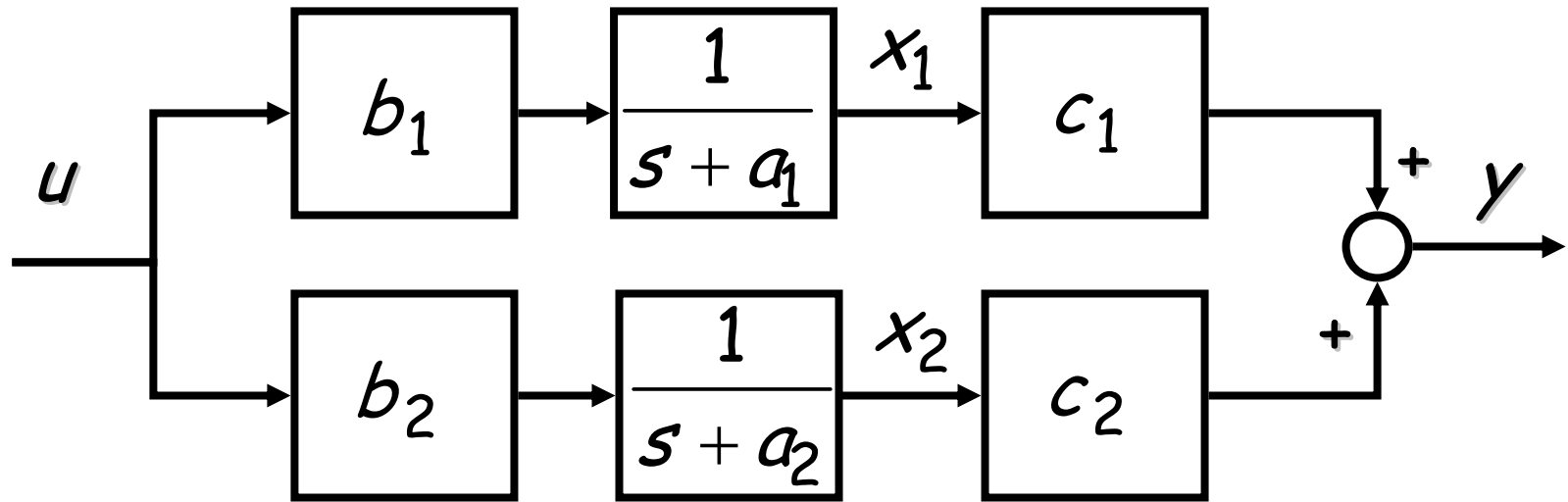
$$\dot{x}_1 = -a_1 x_1 + x_2$$

$$\dot{x}_2 = -a_2 x_2 + u$$

$$y = K' x_1$$

$$\dot{\mathbf{x}} = \begin{pmatrix} -a_1 & 1 \\ 0 & -a_2 \end{pmatrix} \mathbf{x} + \begin{pmatrix} 0 \\ 1 \end{pmatrix} u$$

$$y = (K' \quad 0) \mathbf{x}$$



$$\dot{x}_1 = -a_1 x_1 + b_1 u$$

$$\dot{x}_2 = -a_2 x_2 + b_2 u$$

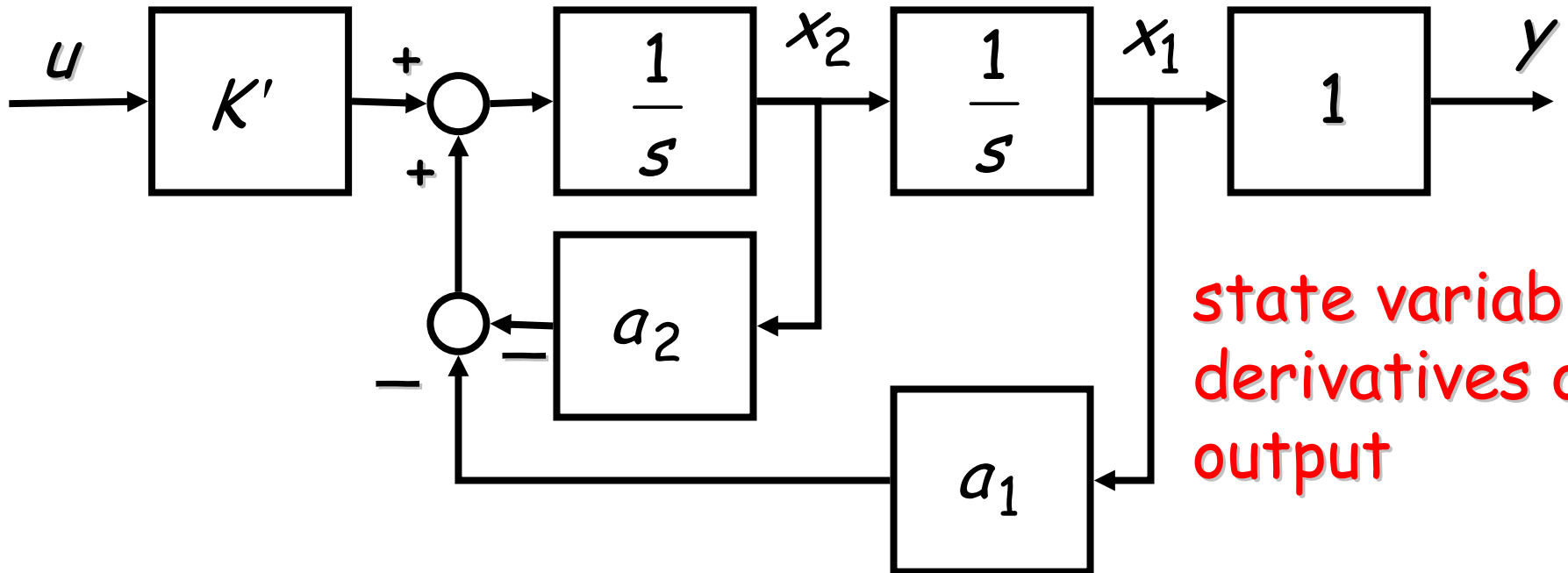
$$y = c_1 x_1 + c_2 x_2$$

$$\dot{\mathbf{x}} = \begin{pmatrix} -a_1 & 0 \\ 0 & -a_2 \end{pmatrix} \mathbf{x} + \begin{pmatrix} b_1 \\ b_2 \end{pmatrix} u$$

$$y = (c_1 \quad c_2) \mathbf{x}$$

eigenvalues at the diagonal

Phase-variable form



state variables
derivatives of
output

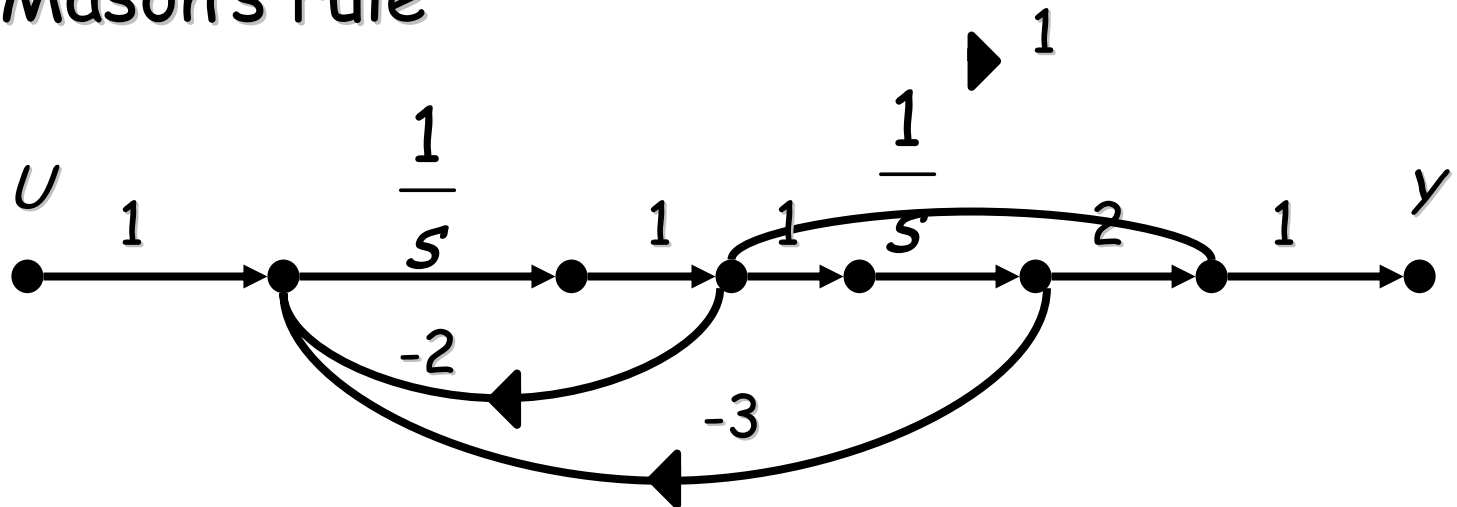
$$\dot{x}_1 = x_2$$
$$\dot{x}_2 = -a_1 x_1 - a_2 x_2 + K' u$$
$$y = x_1$$

$$\dot{\mathbf{x}} = \begin{pmatrix} 0 & 1 \\ -a_1 & -a_2 \end{pmatrix} \mathbf{x} + \begin{pmatrix} 0 \\ K' \end{pmatrix} u$$
$$y = (1 \ 0) \mathbf{x}$$

Phase-variable form (zeros)

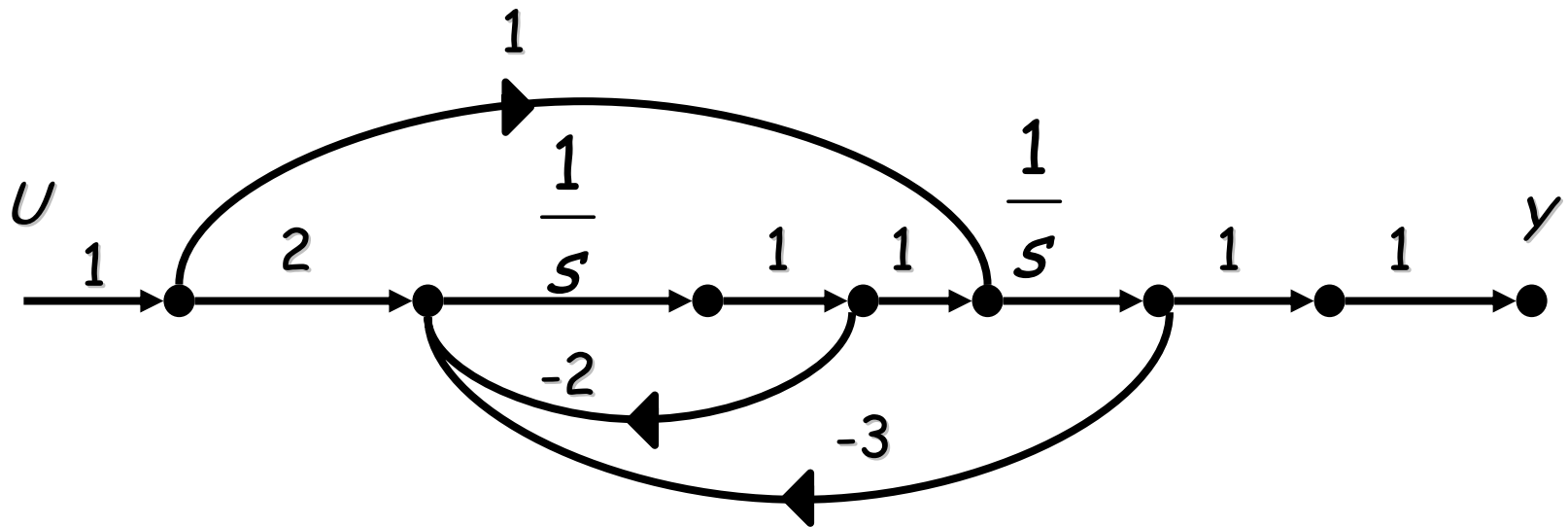
$$\frac{s+2}{s^2+2s+3} = \frac{\frac{1}{s} + \frac{2}{s^2}}{1 + \frac{2}{s} + \frac{3}{s^2}} = \frac{P_1 + P_2}{1 - L_1 - L_2}$$

Mason's rule



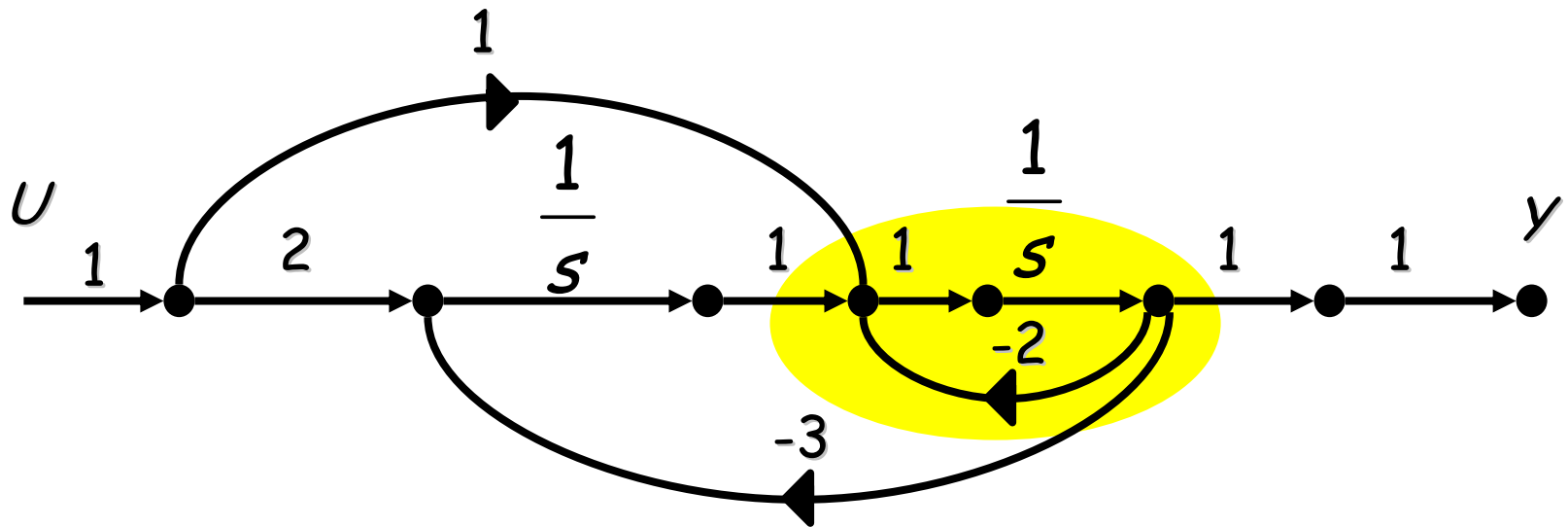
Phase-variable form (alternative)

$$\frac{s+2}{s^2+2s+3} = \frac{1}{s} + \frac{2}{s^2} = \frac{P_1 + P_2}{1 - L_1 - L_2}$$

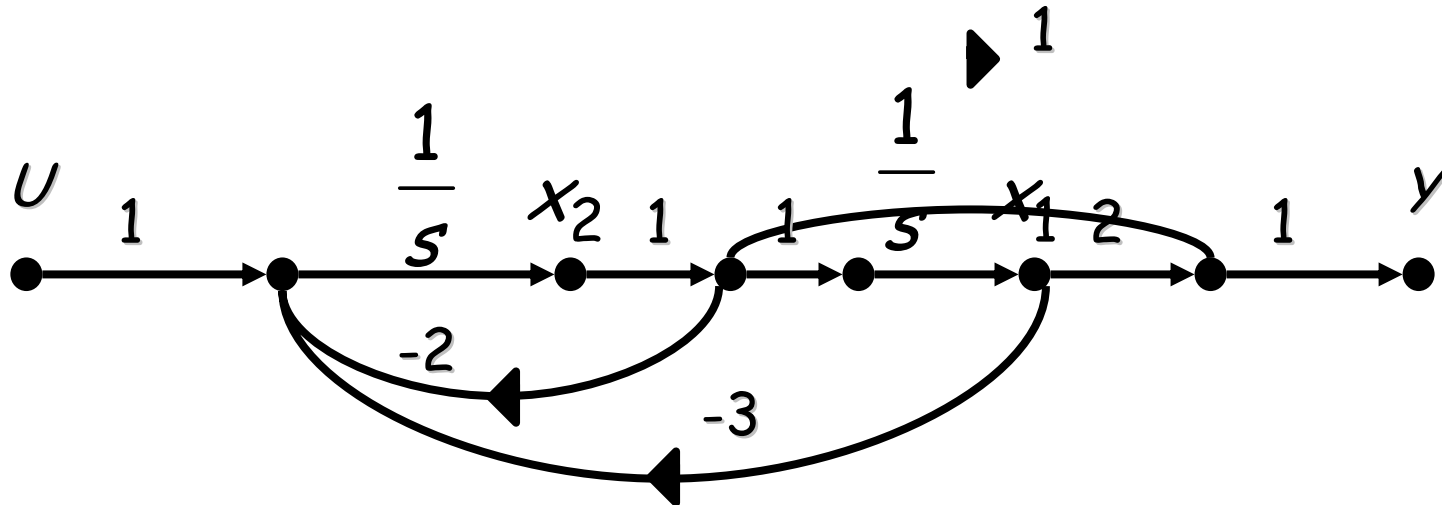


Dual phase-variable form

$$\frac{s+2}{s^2+2s+3} = \frac{1}{s} + \frac{2}{s^2} = \frac{P_1 + P_2}{1 - L_1 - L_2}$$



Phase variable form



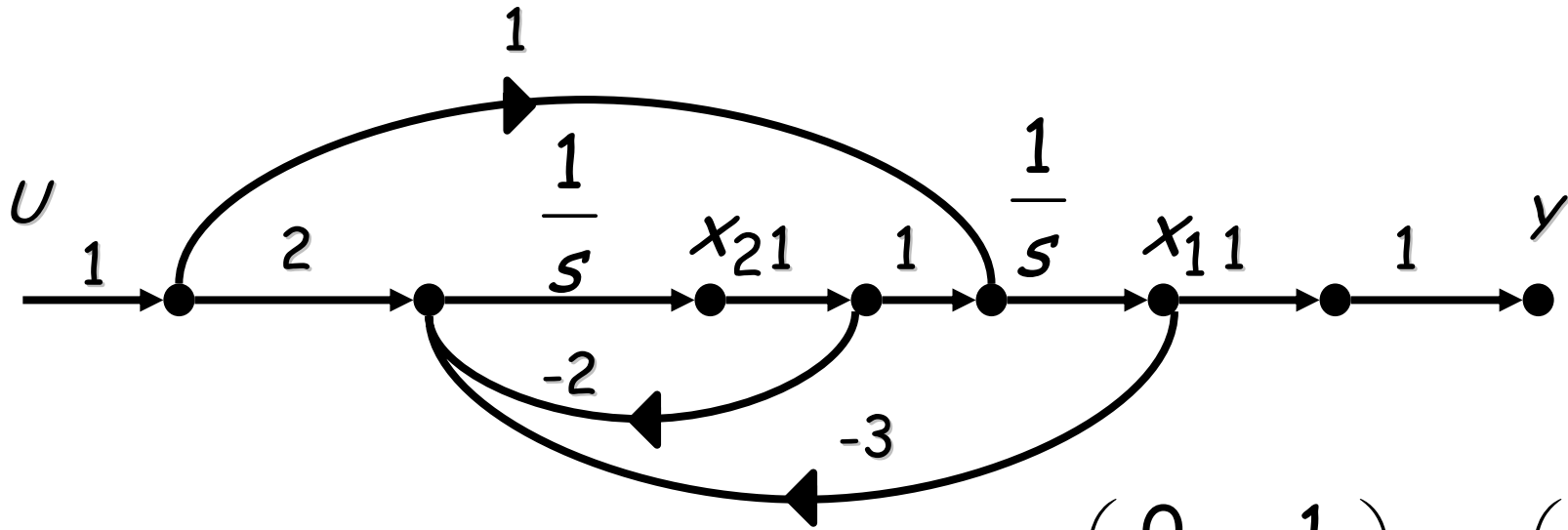
$$A = \begin{pmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ & & & & 1 \\ -a_1 & -a_2 & \dots & \dots & -a_n \end{pmatrix}$$

$$\dot{x} = \begin{pmatrix} 0 & 1 \\ -3 & -2 \end{pmatrix} x + \begin{pmatrix} 0 \\ 1 \end{pmatrix} u$$

$$y = (2 \quad 1) x$$

zero

Phase-variable form (alternative)

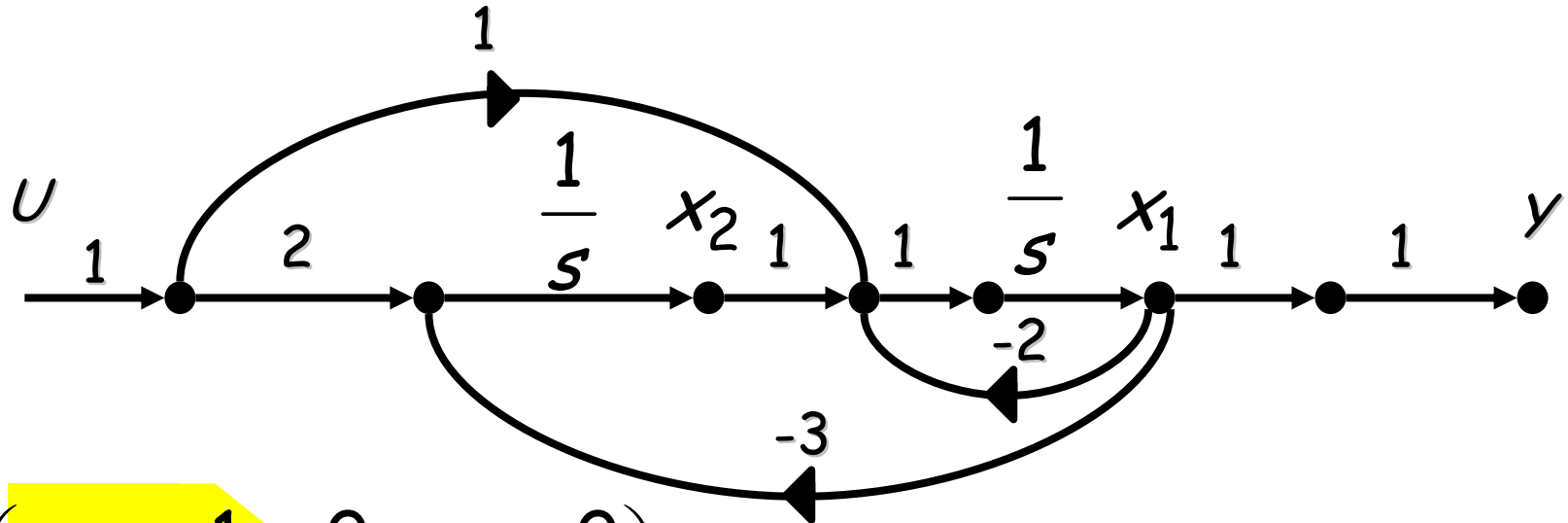


$$\dot{x} = \begin{pmatrix} 0 & 1 \\ -3 & -2 \end{pmatrix} x + \begin{pmatrix} 1 \\ 2 \end{pmatrix} u$$

$$y = (1 \ 0) x$$

zero

Dual phase-variable form



$$A = \begin{pmatrix} -a_1 & 1 & 0 & \dots & 0 \\ -a_2 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ & & & & 1 \\ -a_n & \dots & \dots & \dots & 0 \end{pmatrix}$$

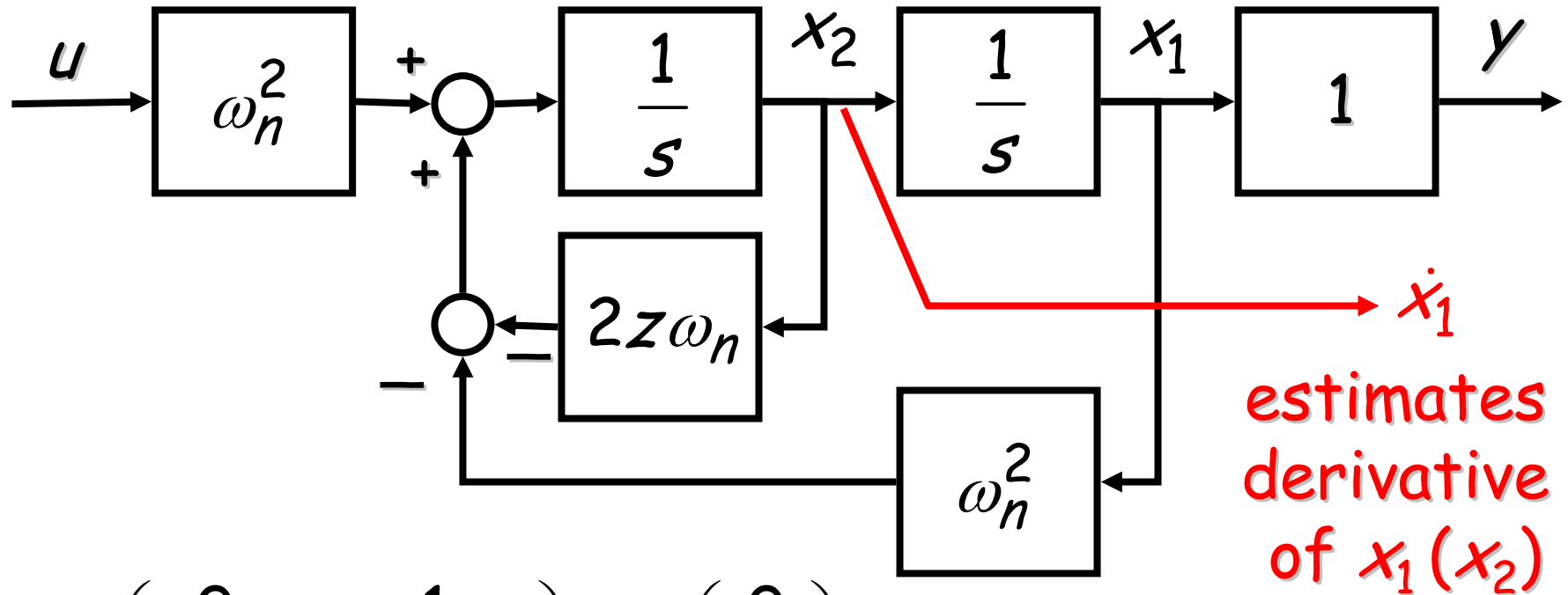
$$\dot{x} = \begin{pmatrix} -2 & 1 \\ -3 & 0 \end{pmatrix} x + \begin{pmatrix} 1 \\ 2 \end{pmatrix} u$$

$$y = (1 \ 0) x$$

zero



State-variable filter



$$\dot{\mathbf{x}} = \begin{pmatrix} 0 & 1 \\ -\omega_n^2 & -2z\omega_n \end{pmatrix} \mathbf{x} + \begin{pmatrix} 0 \\ \omega_n^2 \end{pmatrix} u$$

$$y = (1 \ 0) \mathbf{x}$$

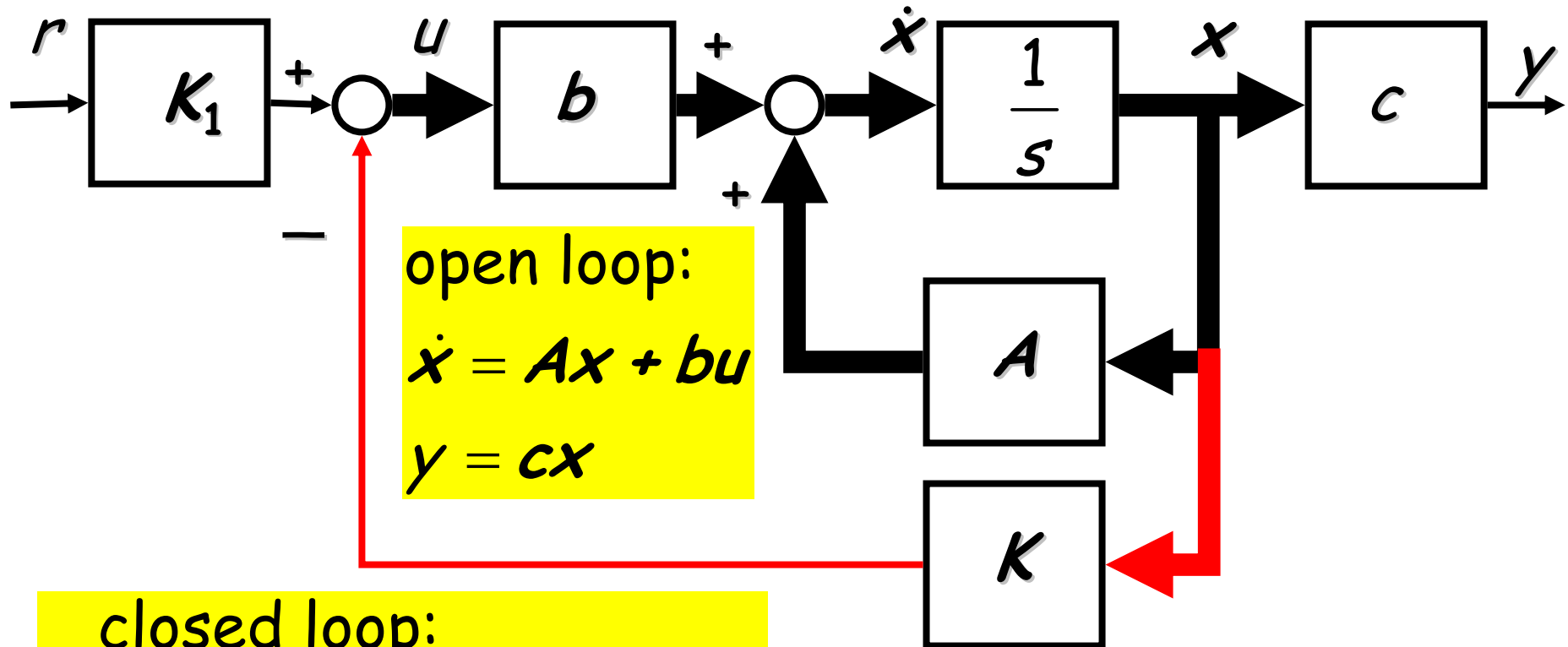
if $y = (0 \ 1) \mathbf{x}$
 $y = \dot{x}_1 \approx \dot{u}$

Demo SVF
bandwidth
Demo_SVF

20-sim

Demo SVF
noise
Demo_SVF_noise

State space design



closed loop:

$$\dot{x} = (A - bK)x + bK_1r$$

$$y = cx$$

$$\dot{x} = (A - bK)x + K_1 br$$

$$A' = (A - bK)$$

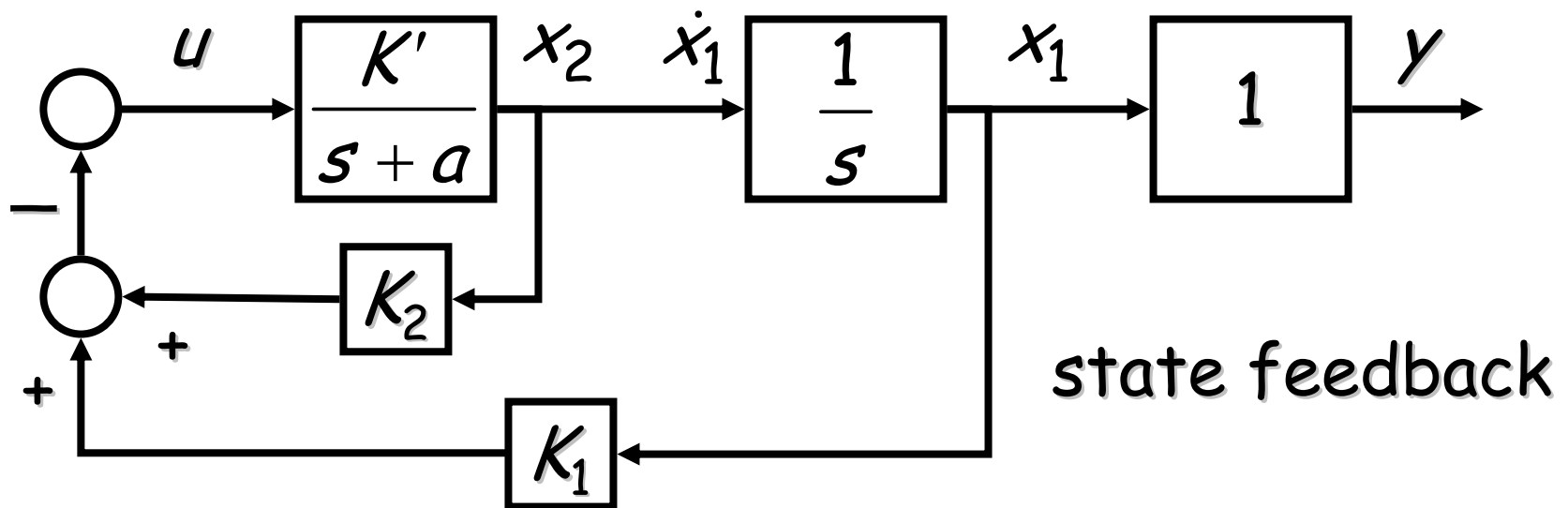
When K is properly chosen,
 A' can get any desired eigen values

Poles can be placed by means of
state feedback

(stable) zeros can only be relocated
by means of prefilter

Example

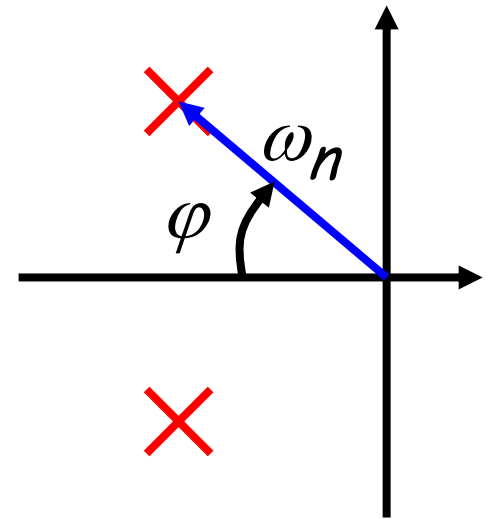
Consider the process: $\dot{x} = \begin{pmatrix} 0 & 1 \\ 0 & -a \end{pmatrix} x + \begin{pmatrix} 0 \\ K' \end{pmatrix} u$



$$A' = \begin{pmatrix} 0 & 1 \\ -K'K_1 & -a - K'K_2 \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -\omega_n^2 & -2z\omega_n \end{pmatrix}$$

Example

$$A' = \begin{pmatrix} 0 & 1 \\ -K'K_1 & -a - K'K_2 \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -\omega_n^2 & -2z\omega_n \end{pmatrix}$$



$$K'K_1 = \omega_n^2$$

$$K'K_2 = 2z\omega_n - a$$

$$a + K'K_2 = 2z\omega_n$$

$$= 2z\sqrt{K'K_1} - a$$

$$z = \cos(\varphi)$$

if $K' = 1, a = 1$ $\omega_n = 2, z = 0.7$

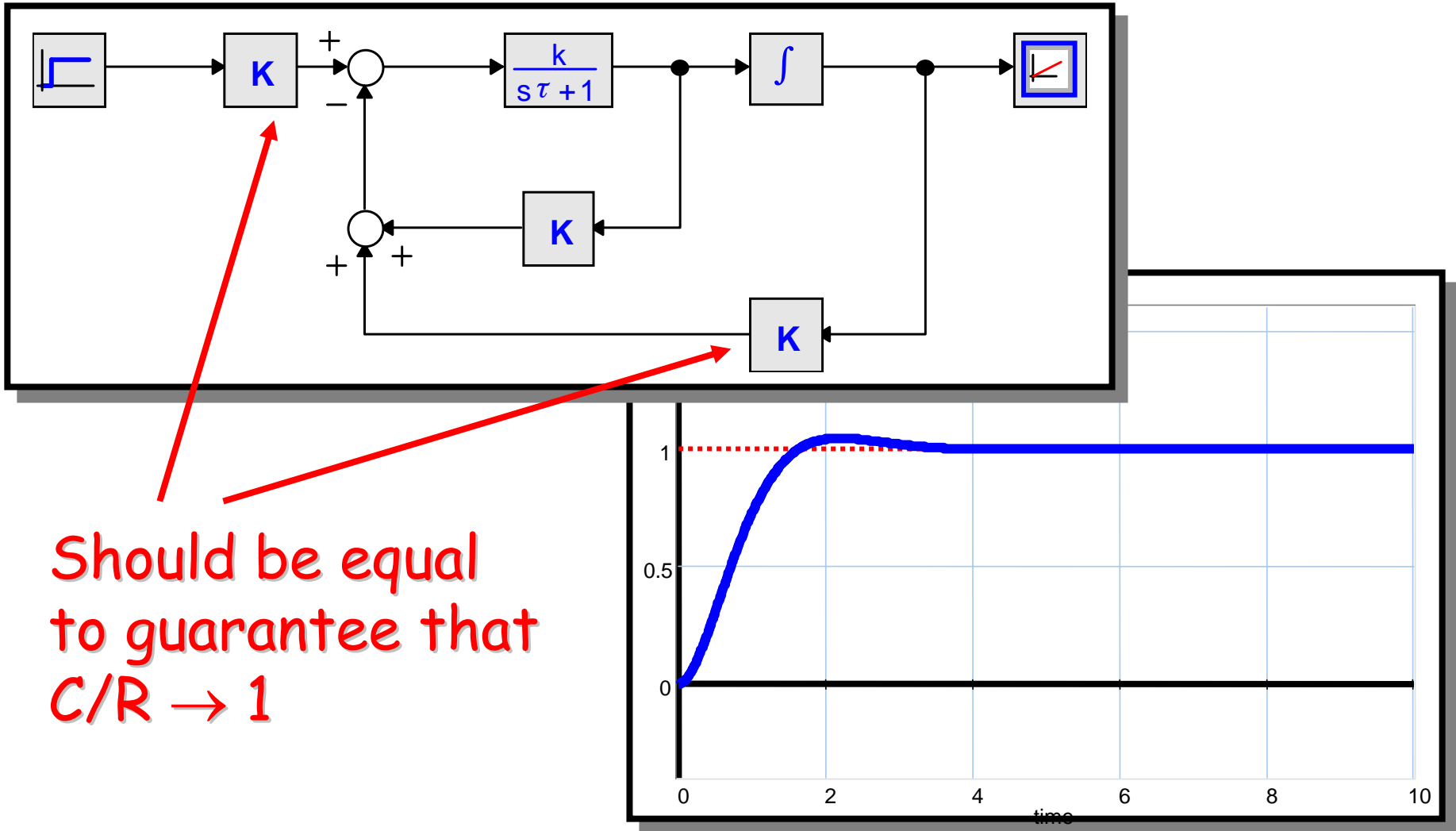
process

$$K_1 = 2^2 = 4$$

$$K_2 = 2z2 - 1 = 1.8$$

Design choice

Simulation

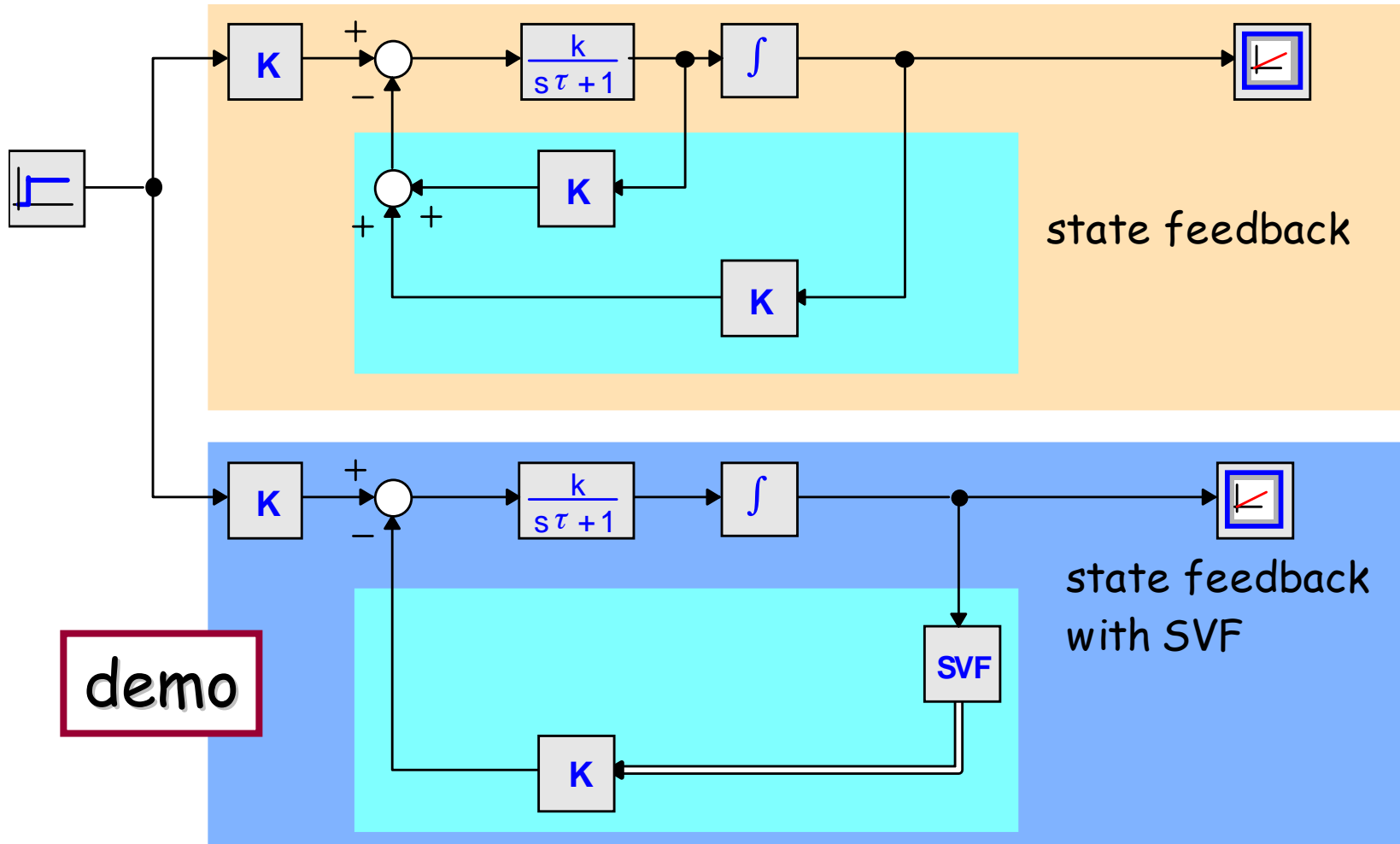


- State feedback assumes that all states can be used for feedback...
- This implies that

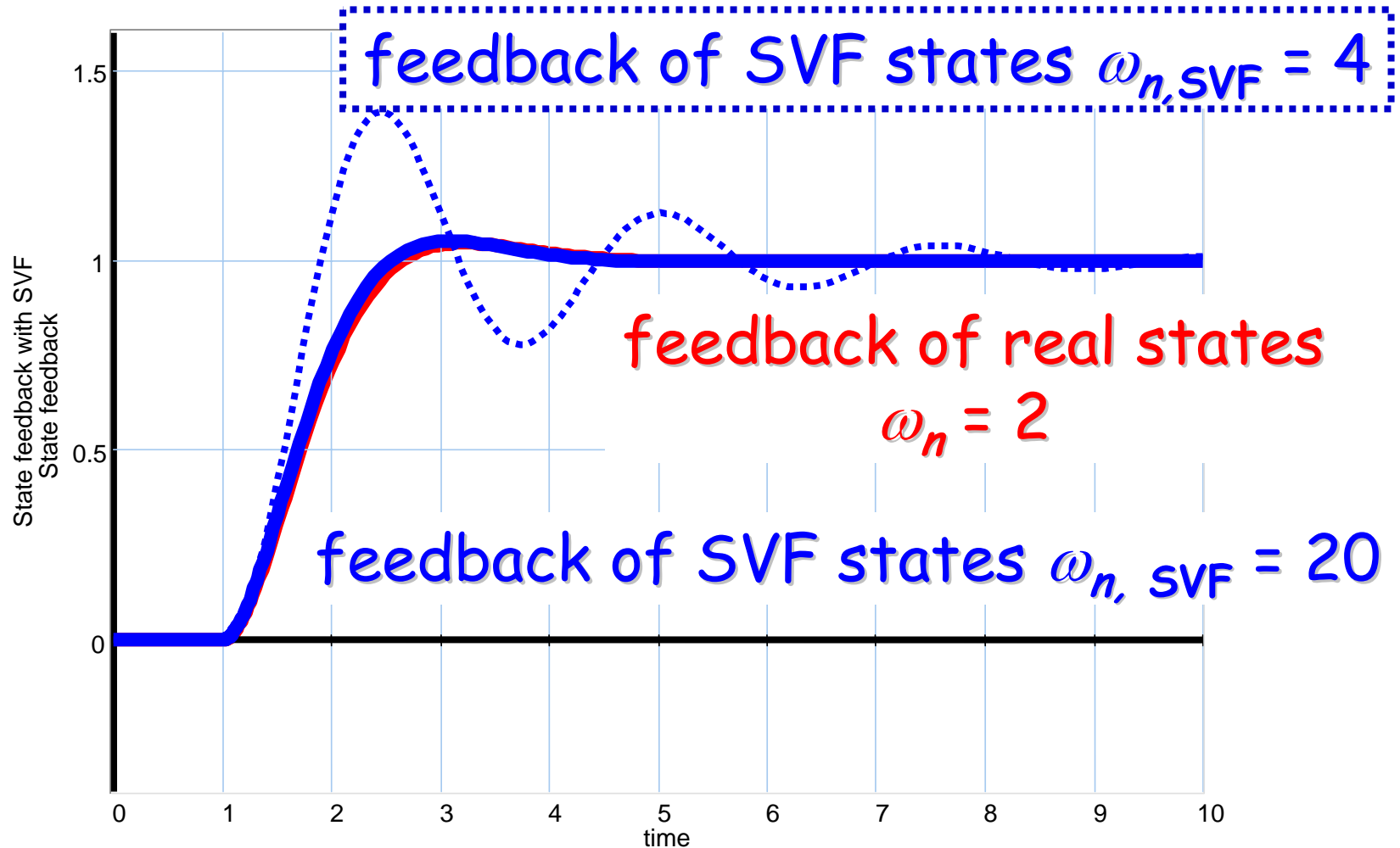
$$y = \begin{pmatrix} y_1 \\ y_2 \end{pmatrix} = \begin{pmatrix} 1 & 0 \\ 0 & 1 \end{pmatrix} x$$

- If all states are not available they can be estimated
 - e.g. with a state variable filter (SVF)

Demonstration 20-sim



Responses



- If the bandwidth of the SVF is chosen 10 times larger than the bandwidth of the controlled process, the phase lag of the SVF is negligible.
- Can only be done when there is (almost) no noise on measured y
- Course 'Digital Control' will give more advanced solutions

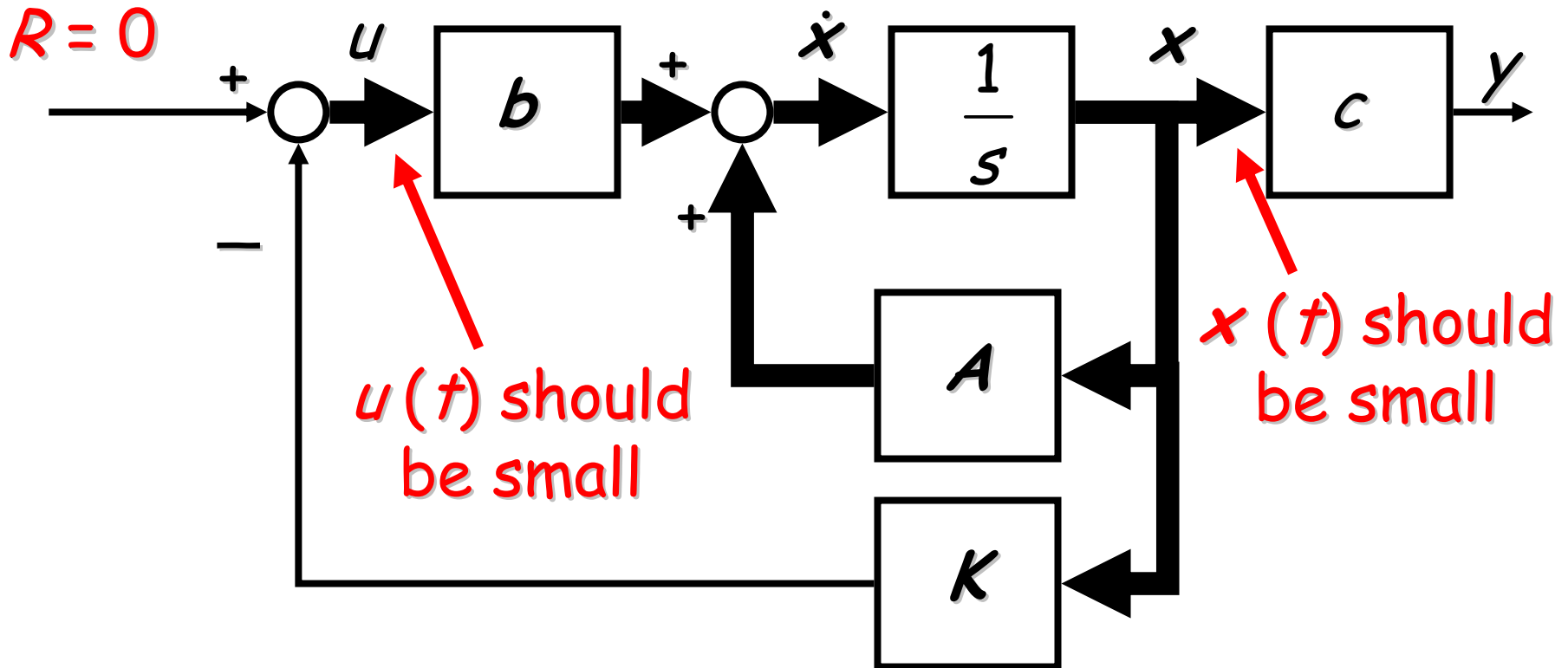
- Performance of a system can be expressed in terms of
 - bandwidth
 - pole locations (in fact the same)
 - optimal control problem

- Error should be small
- reference changes should be perfectly tracked

But

- not at any price:
- control effort should be kept small
 - energy
 - price of equipment

Regulator system



Consider errors at $t = 0$ ($x(0) \neq 0$)

We consider the system

$$\dot{x} = Ax + bu, \text{ with state feedback}$$

$$u = -Kx$$

System description

Find the feedback gain, K , such that

Adjustable parameter(s)

$$J = \int_0^{\infty} (x^T Qx + ru^2) dt \text{ is minimal}$$

Criterion

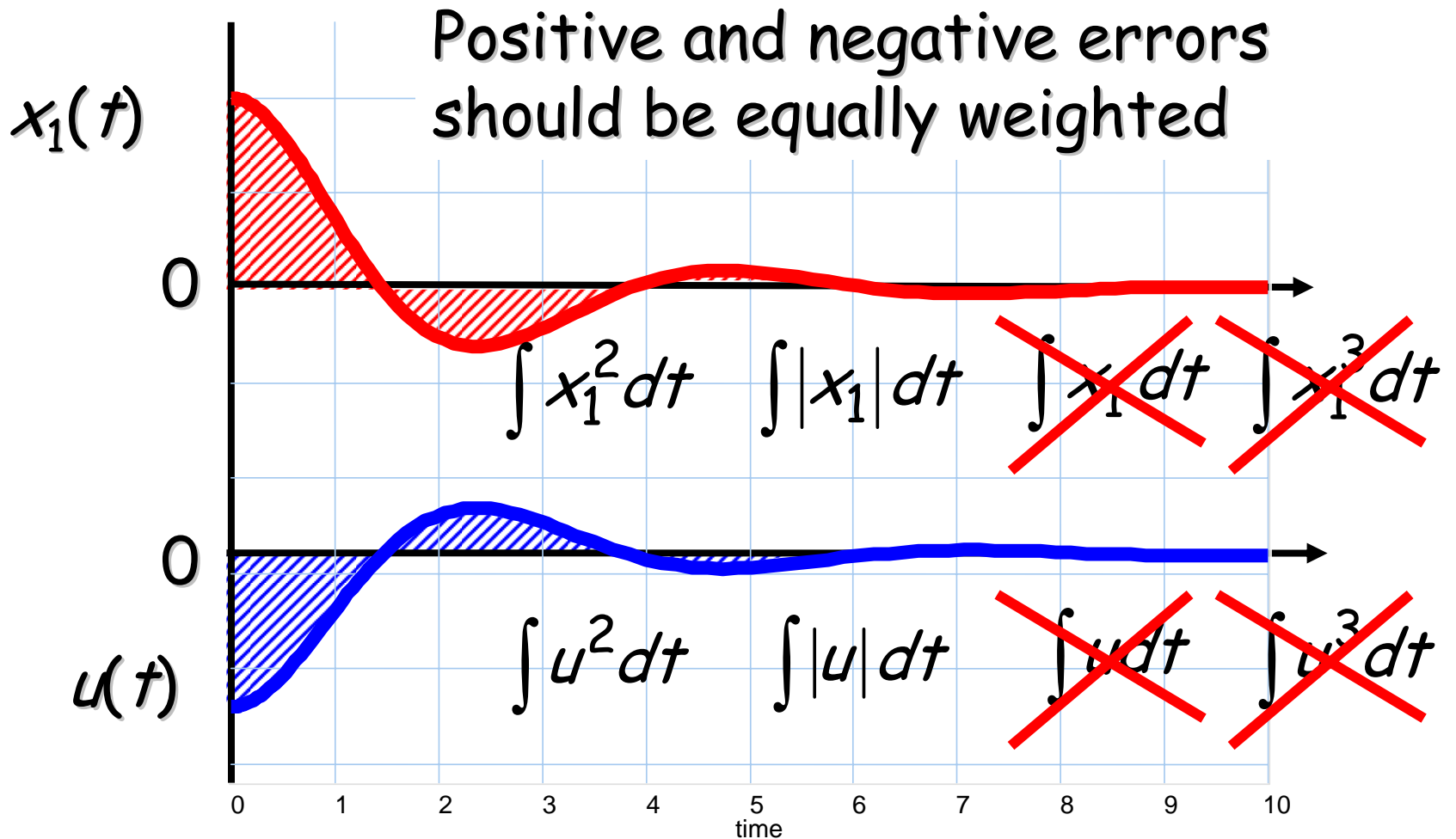
Quadratic criterion

If we consider a second-order system and

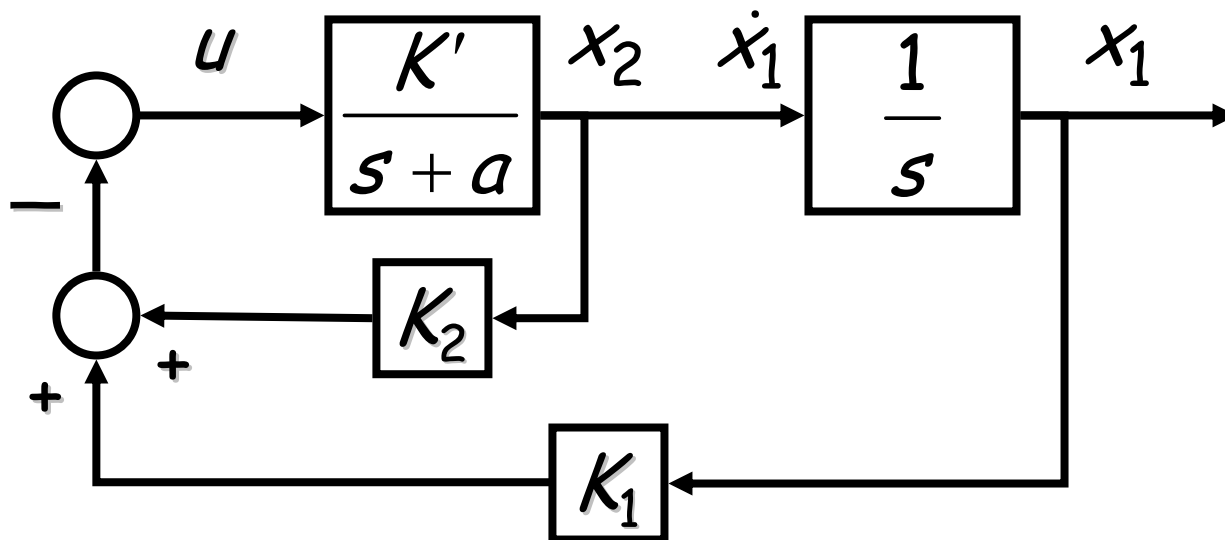
$$Q = \begin{pmatrix} 1 & 0 \\ 0 & 0 \end{pmatrix}$$

then
$$J = \int_0^{\infty} (x^2 + ru^2) dt$$

Meaningful criteria



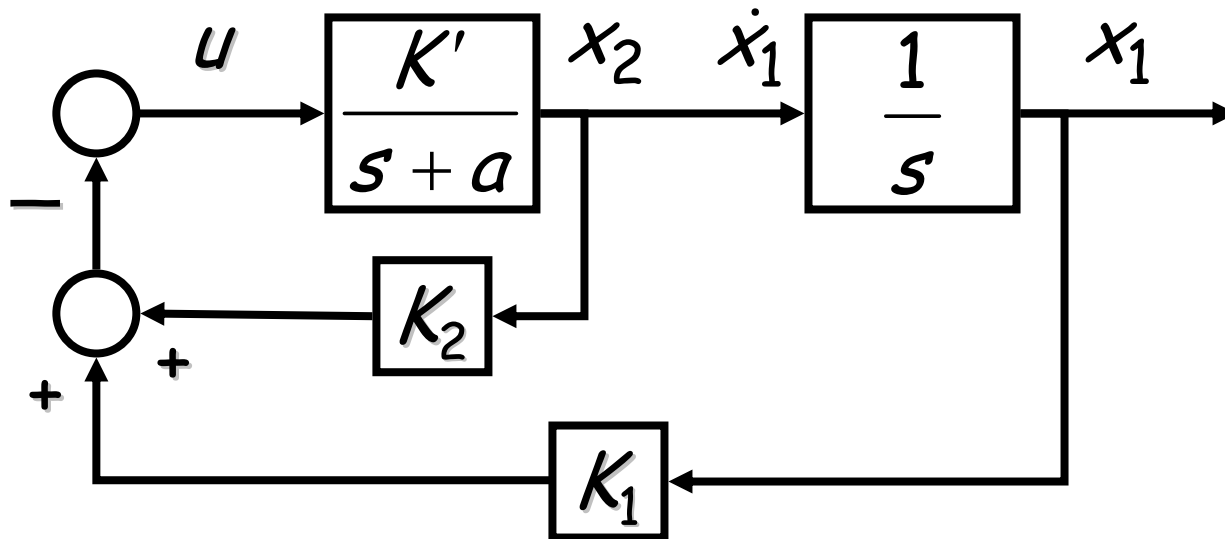
Not properly defined optimisation problem



Find the feedback gains, K_1 , K_2 , such that $\int x_1^2 dt$ is minimal

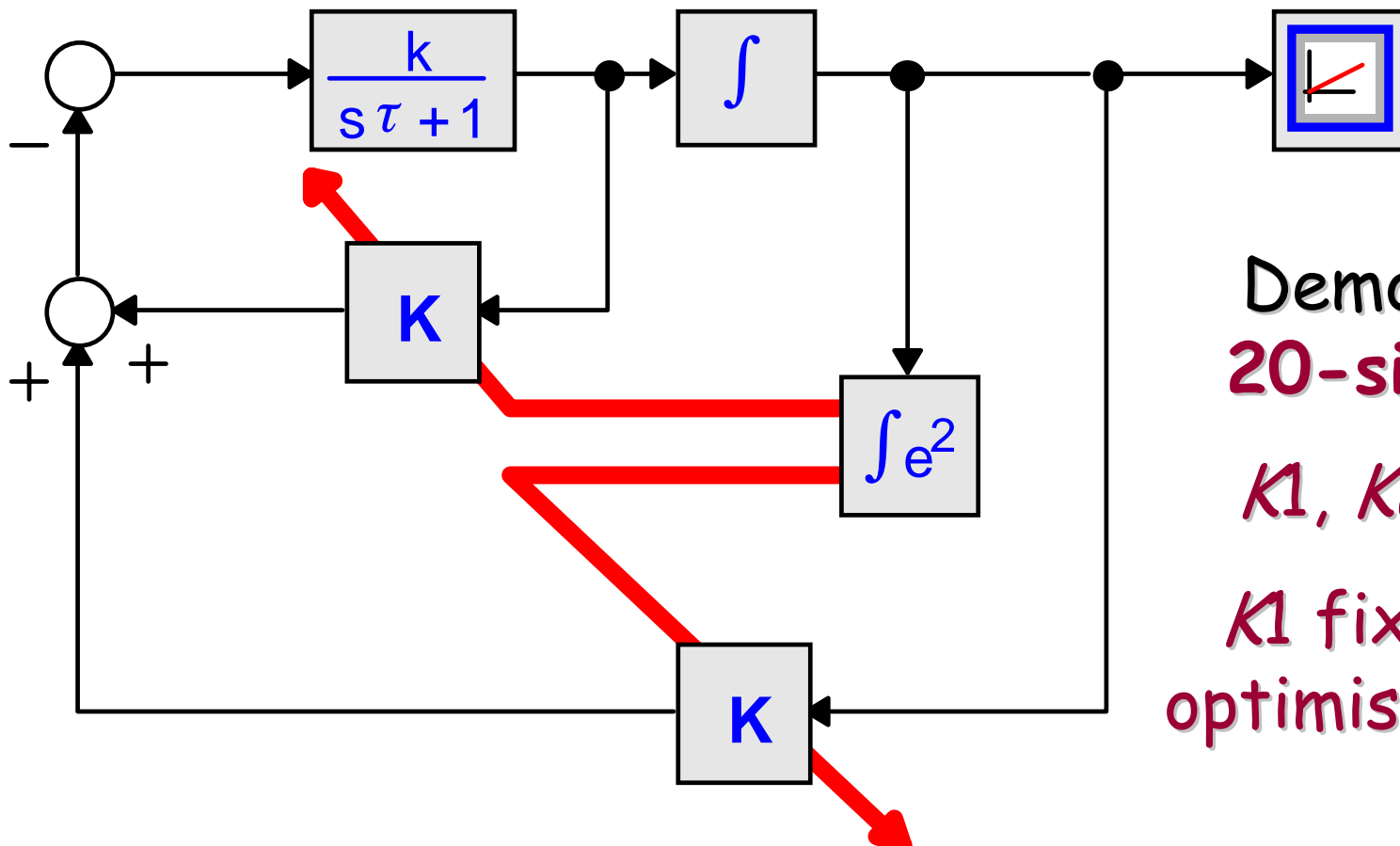
K_1, K_2 go to ∞

Properly defined optimisation problem



Given K_1 , find the feedback gain, K_2 , such that $\int x_1^2 dt$ is minimal

- Based on Ricatti equations
 - LQR in 20-sim or Matlab
 - only for quadratic criteria
- Hill climbing
 - systematic search method
 - e.g. 20-sim
 - any well chosen criterion
- Hill climbing
 - find the top of an unknown hill in the fog



Demo's
20-sim:
K1, K2
K1 fixed
optimise K2

Demo: state feedback optimization K1 fixed

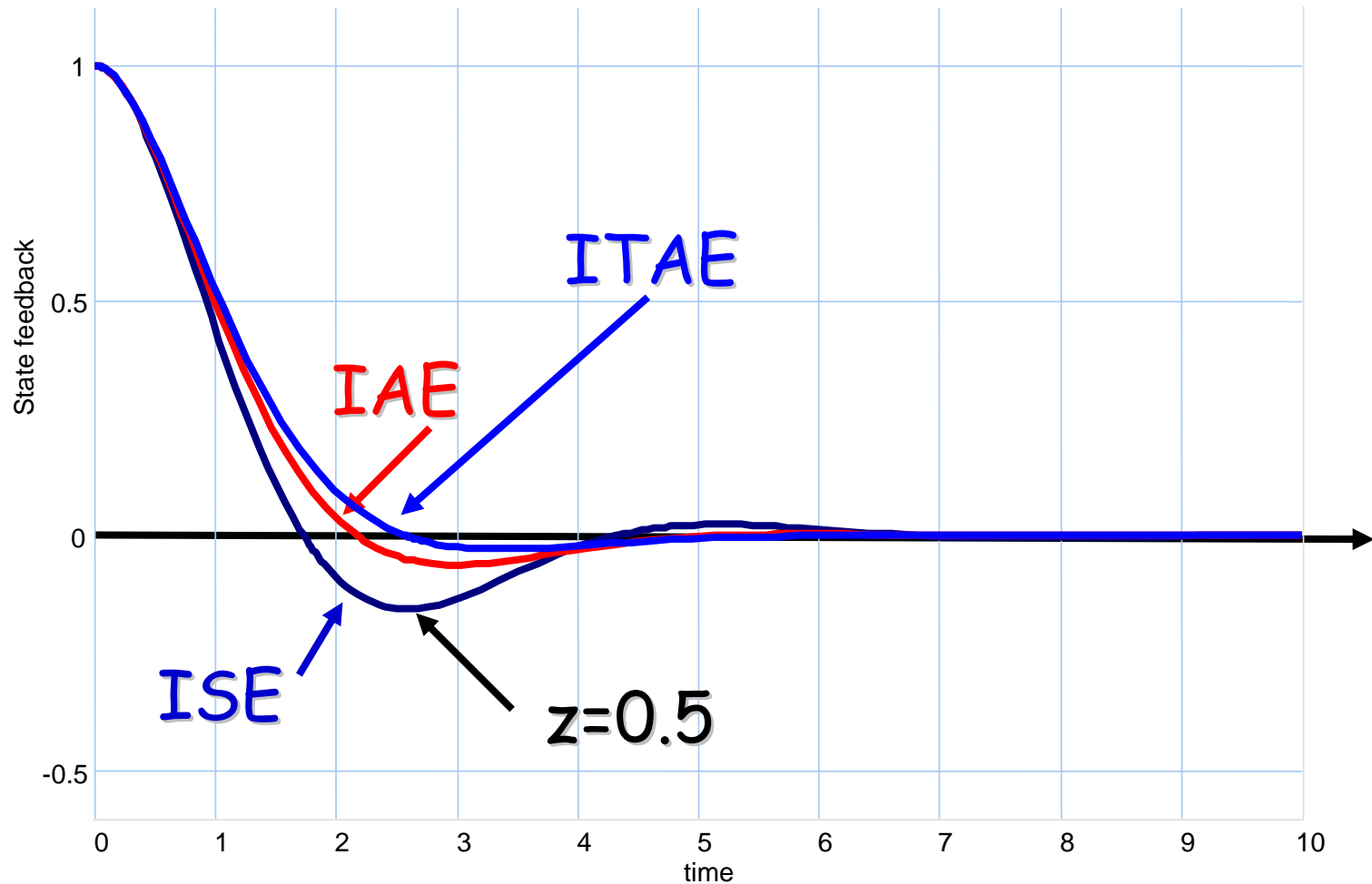
Demo: state feedback optimization

ISE $\int e^2 dt$ more weight
of large errors

IAE $\int |e| dt$

ITAE $\int |e| t dt$ more weight
on steady-state
errors

Responses

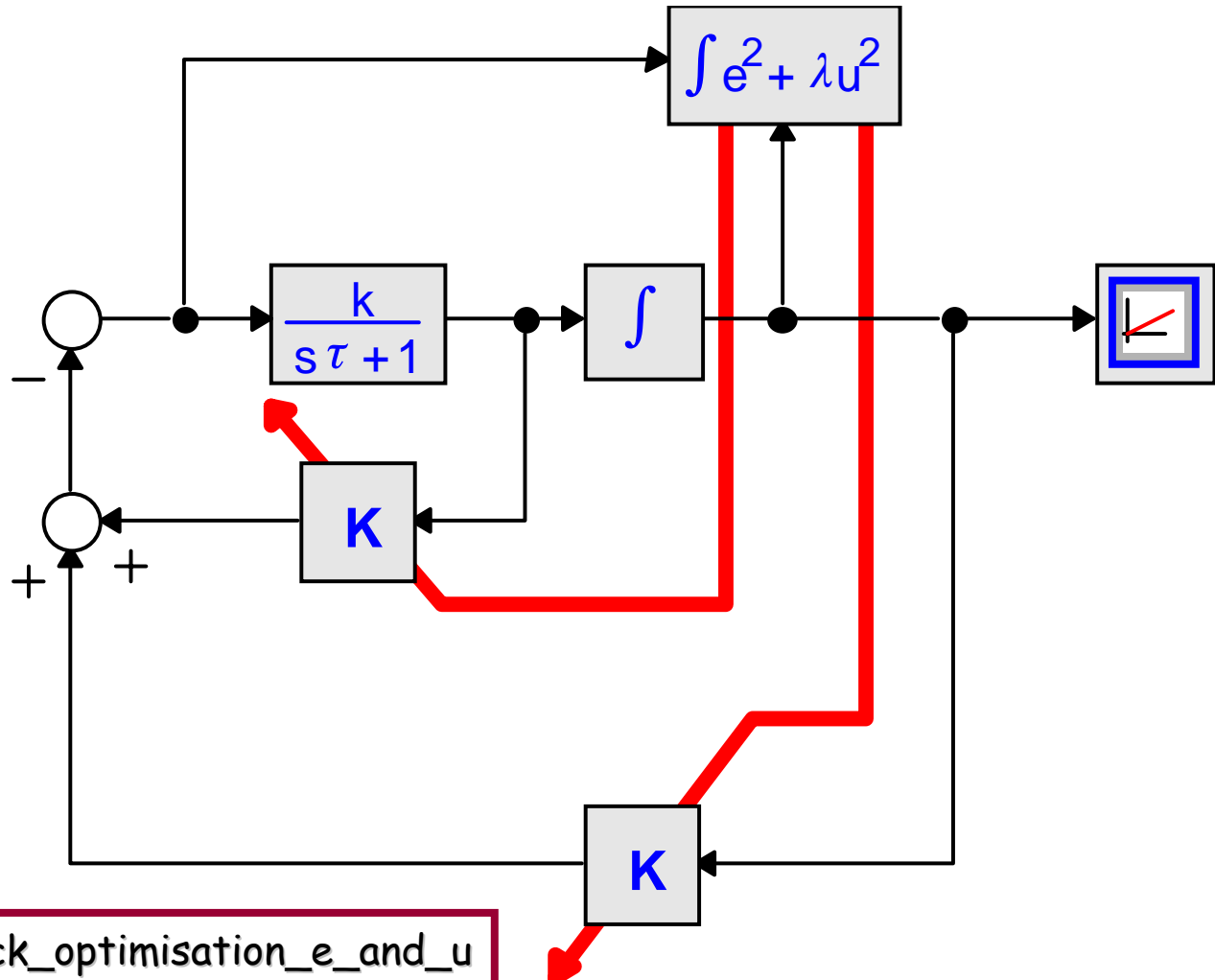


- When we weight both x and u , all feedback gains may be optimised simultaneously

$$J = \int (e^2 + \lambda u^2) dt$$

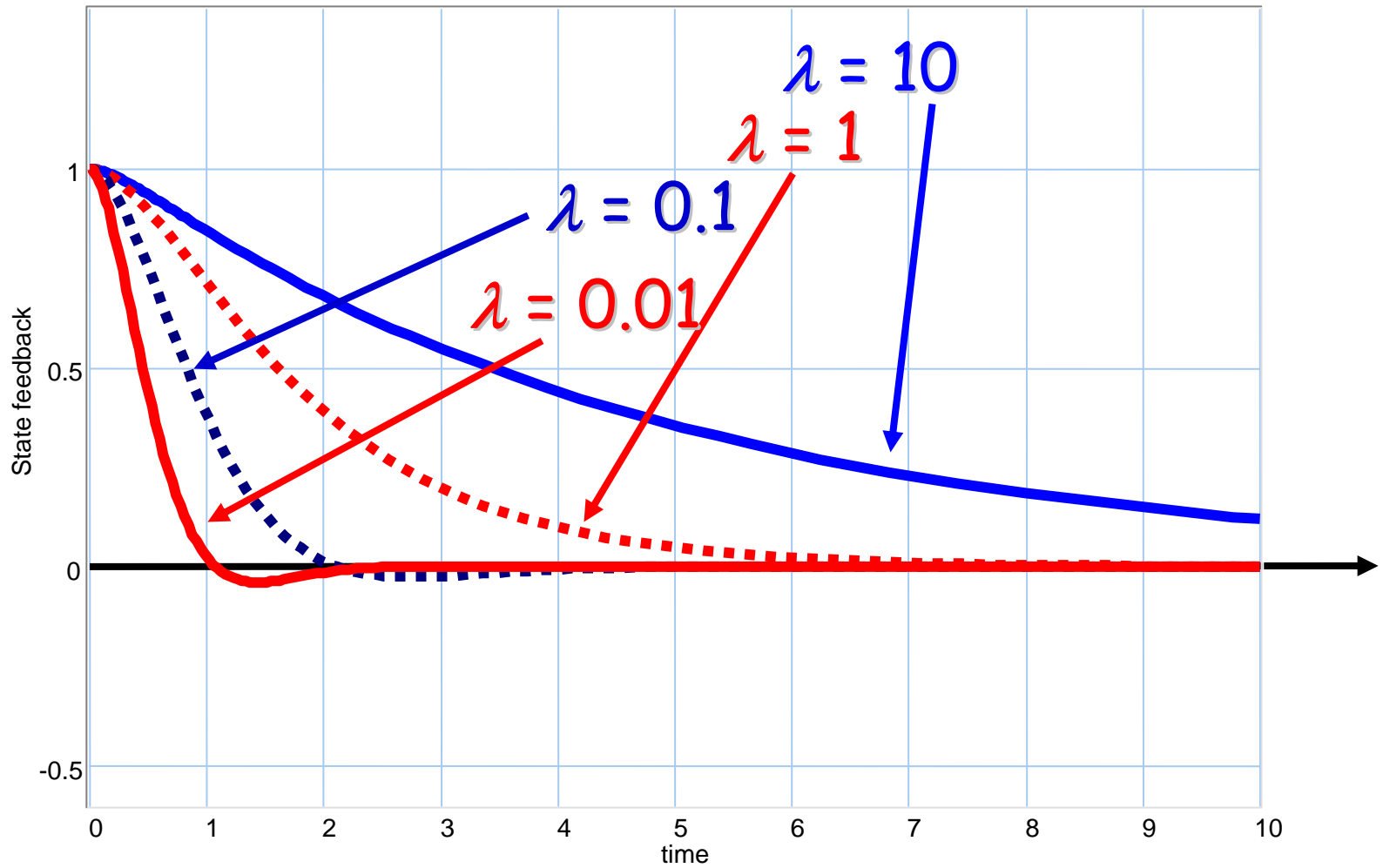
more weight on λ , leads to smaller u , and slower response

Weighting x and u

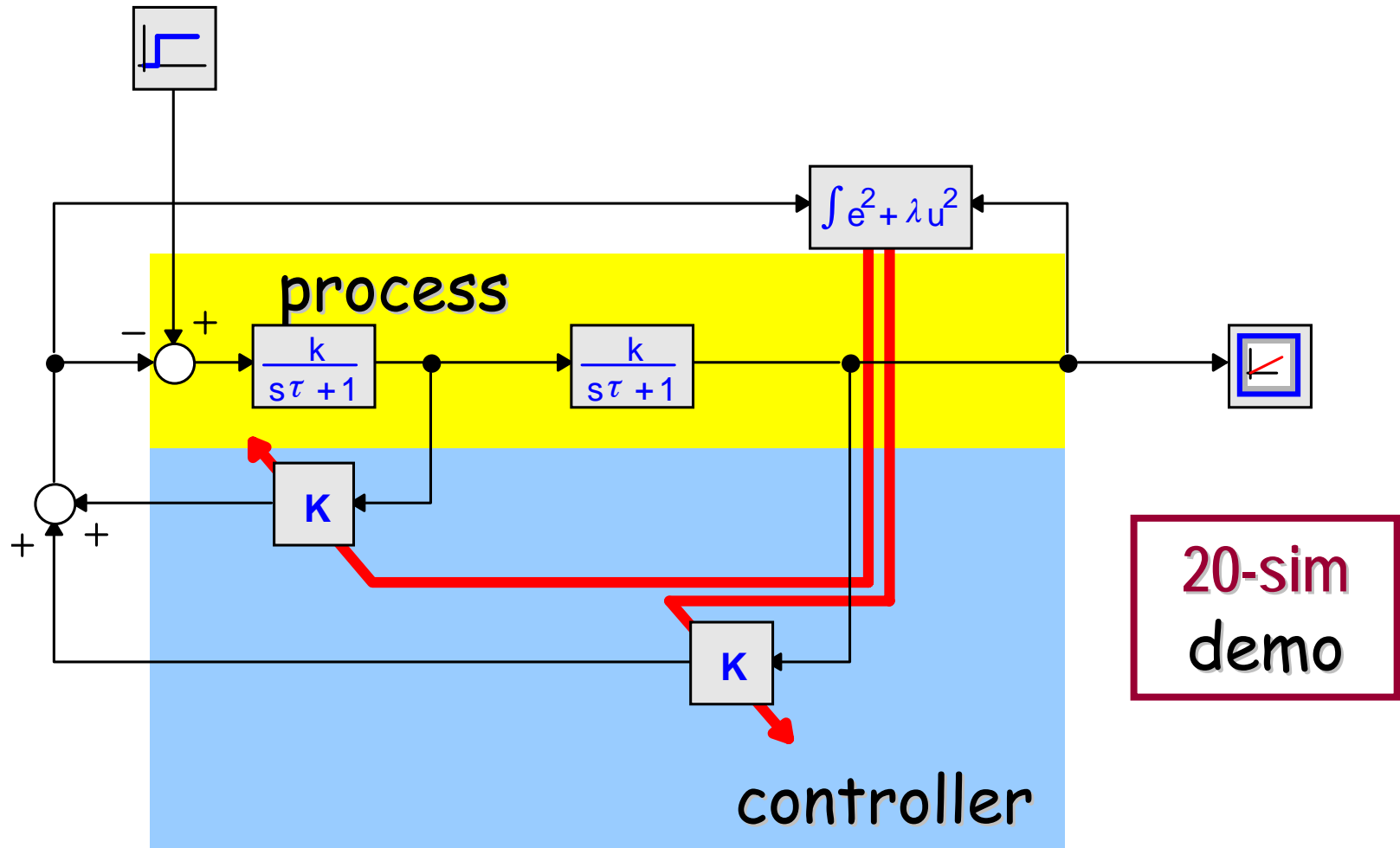


Demo: State_feedback_optimisation_e_and_u

Responses

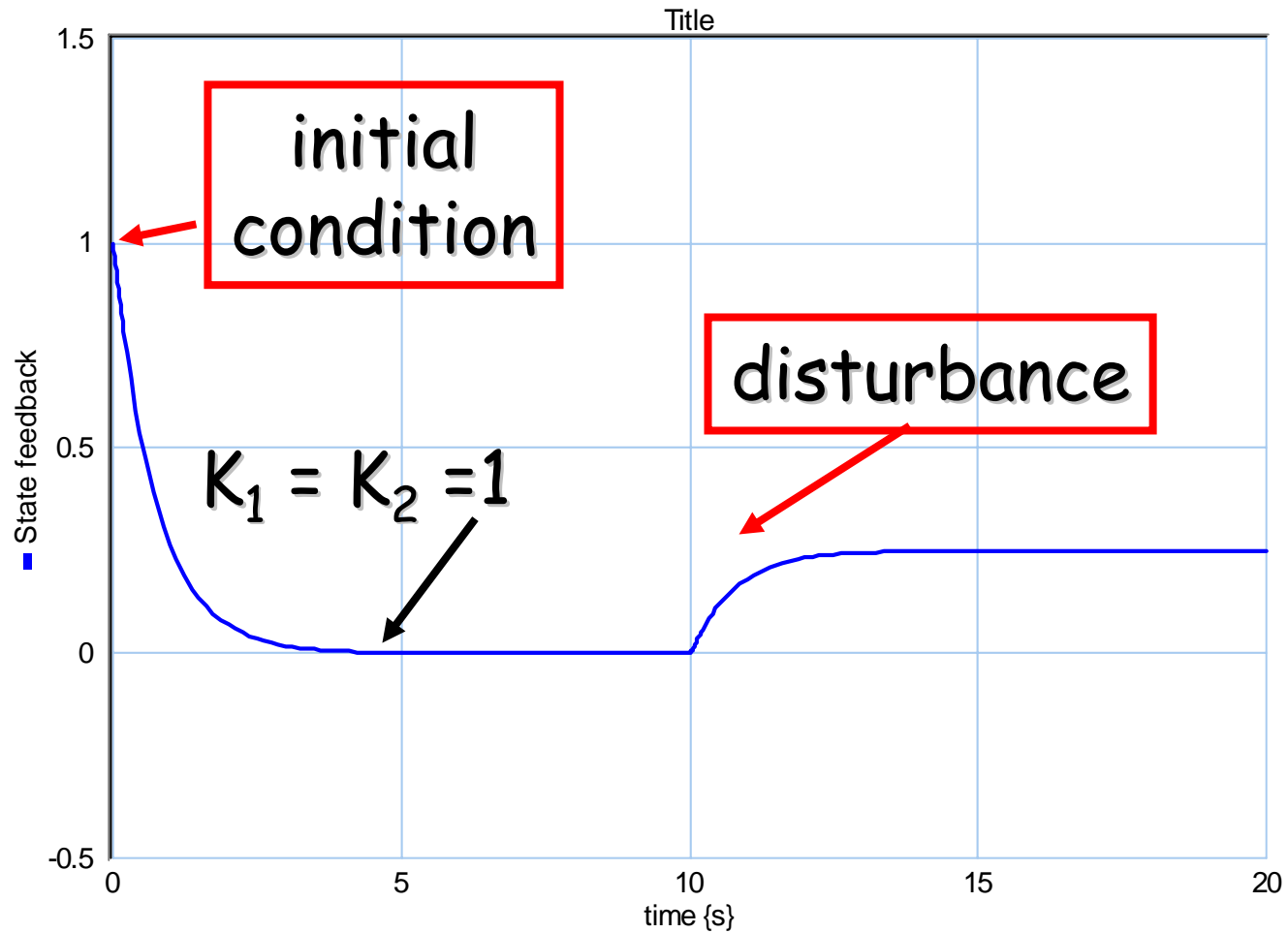


Type 0 system

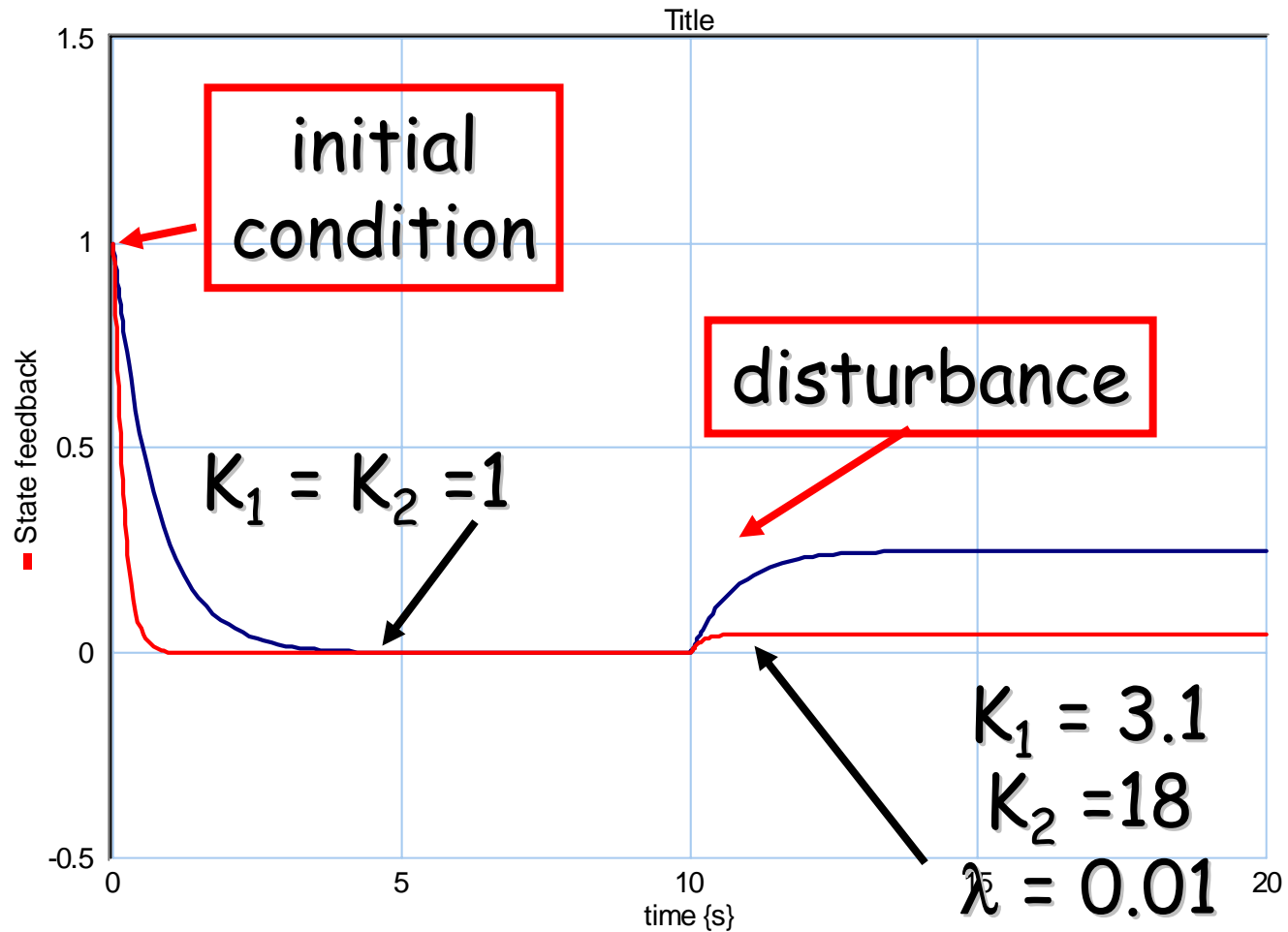


20-sim
demo

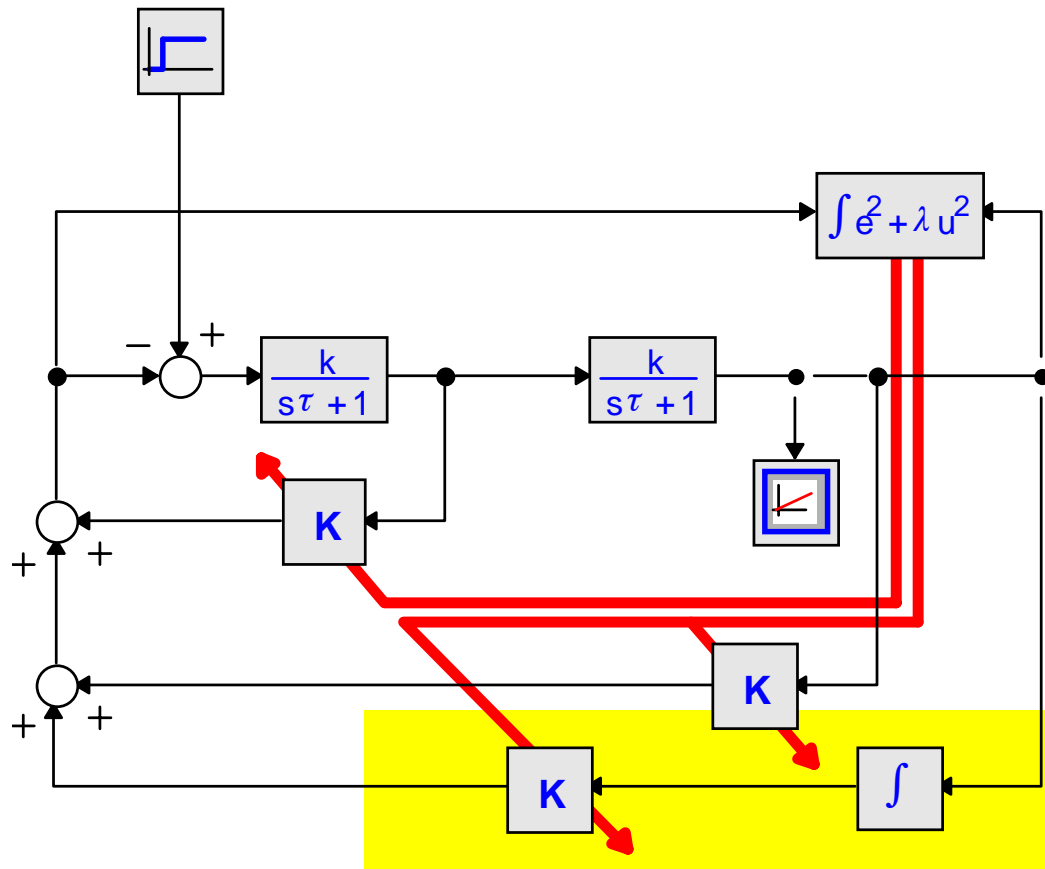
Responses



Responses



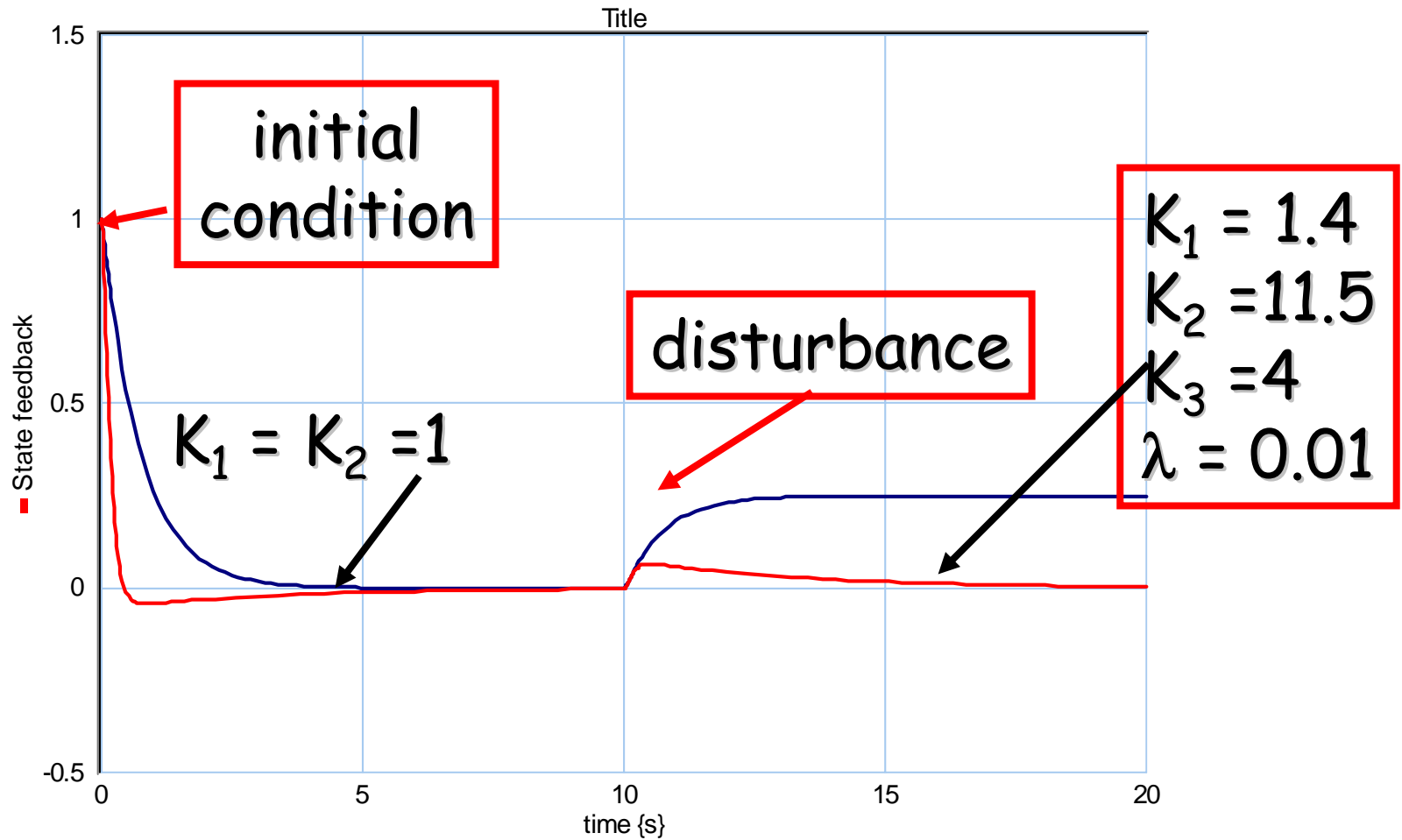
'PID'-control



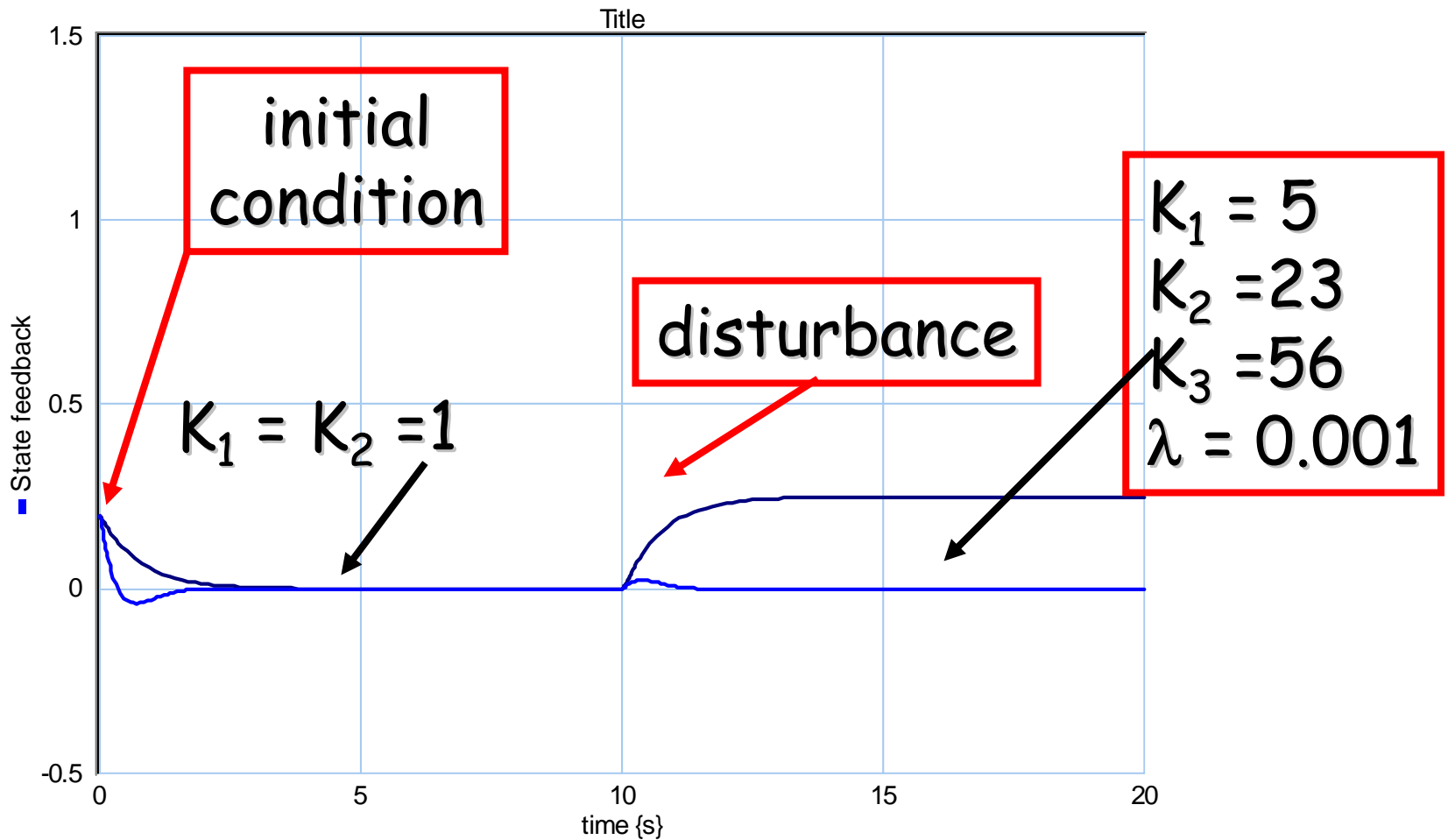
integral control

20-sim
demo

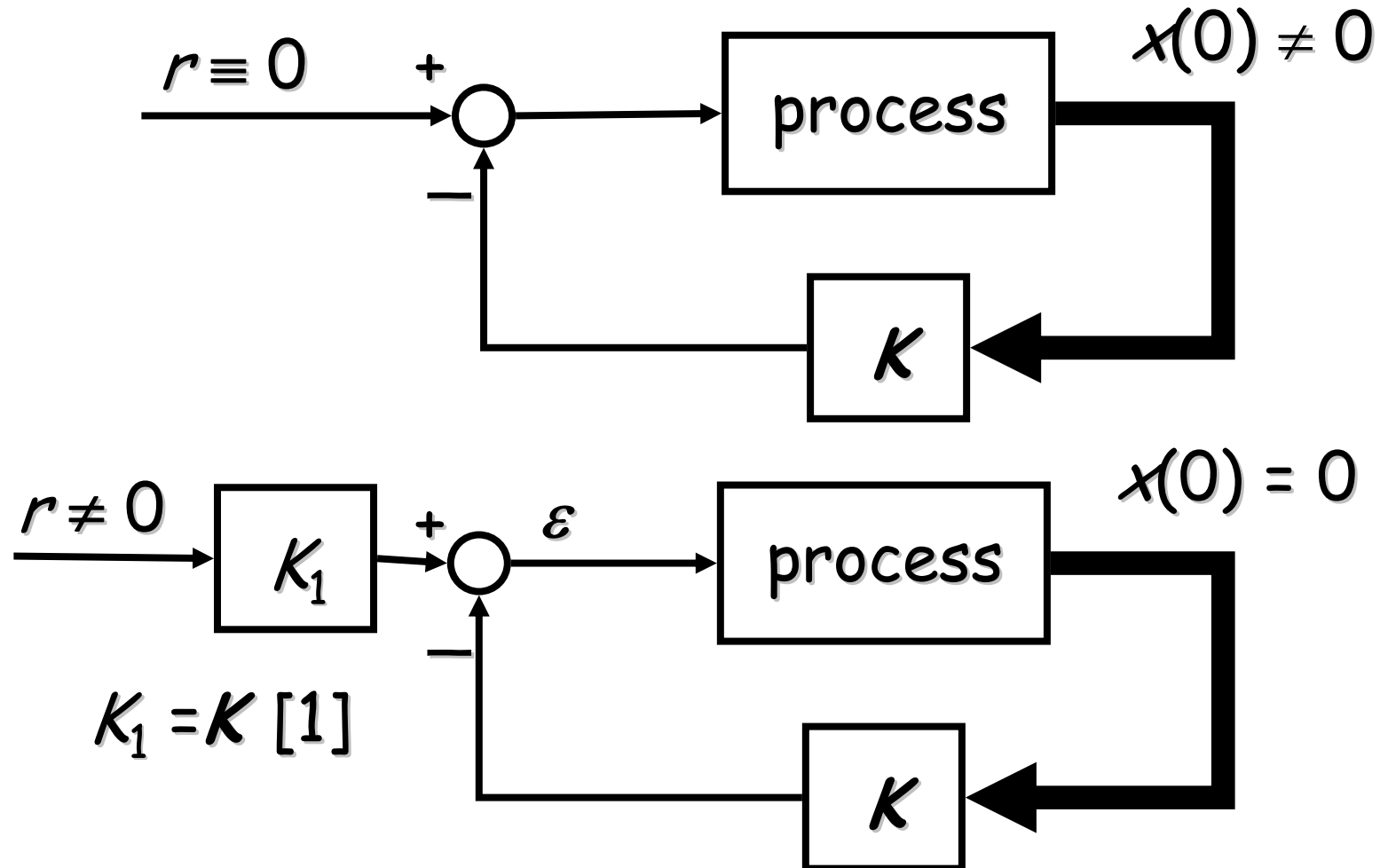
Responses



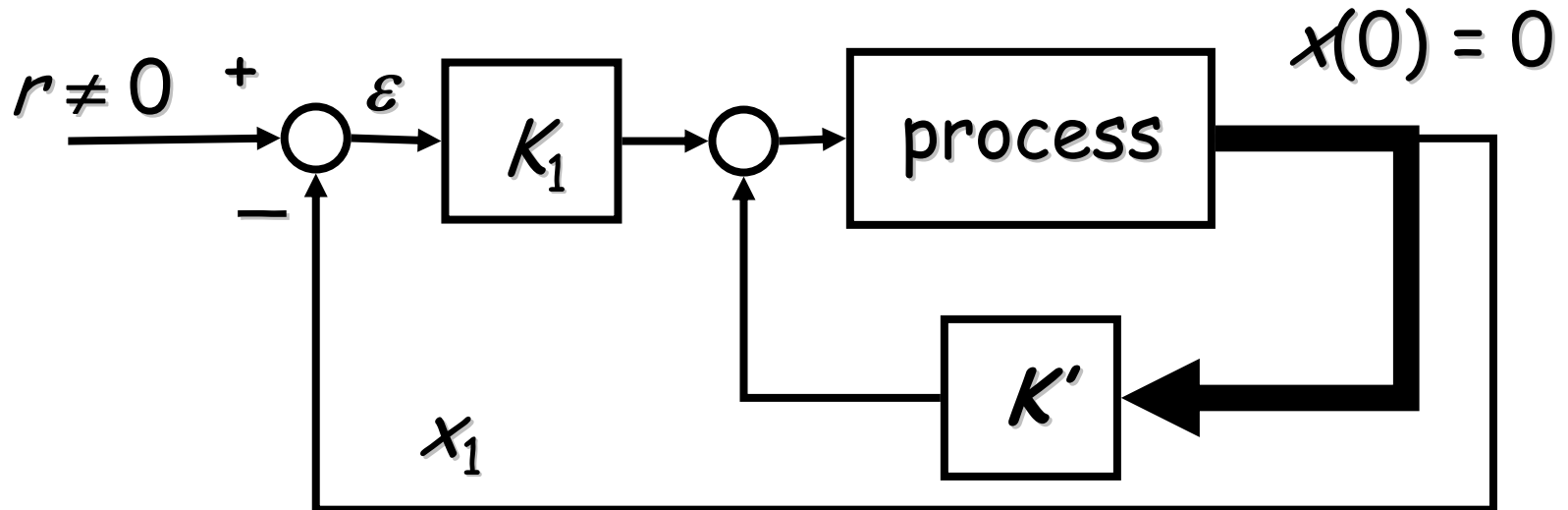
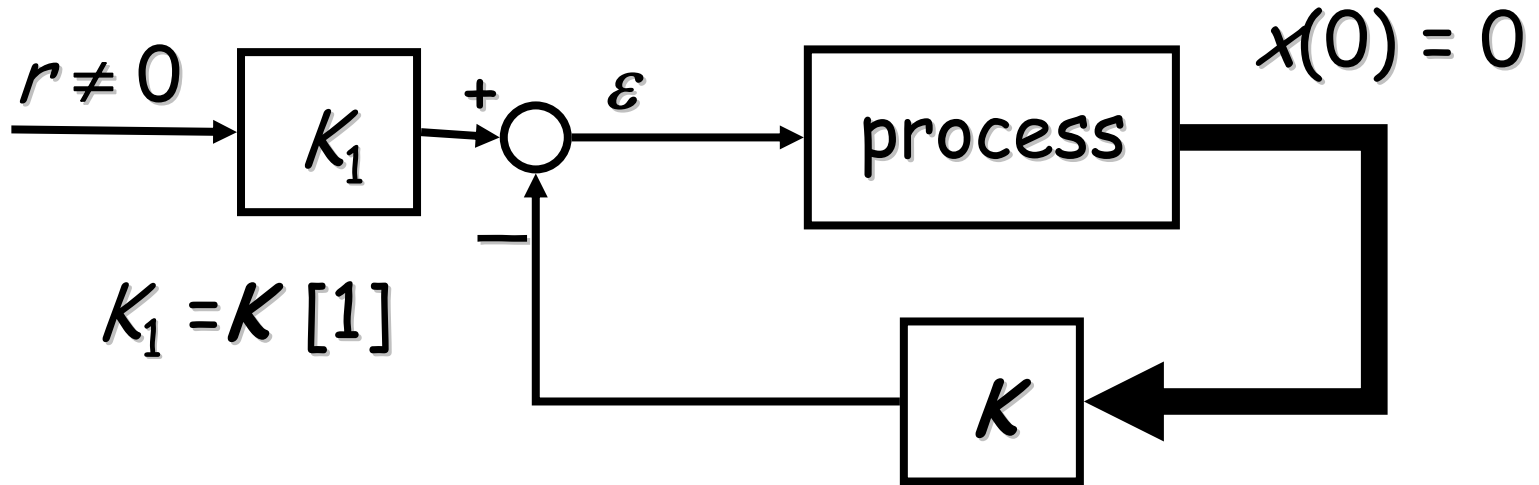
Responses (2)



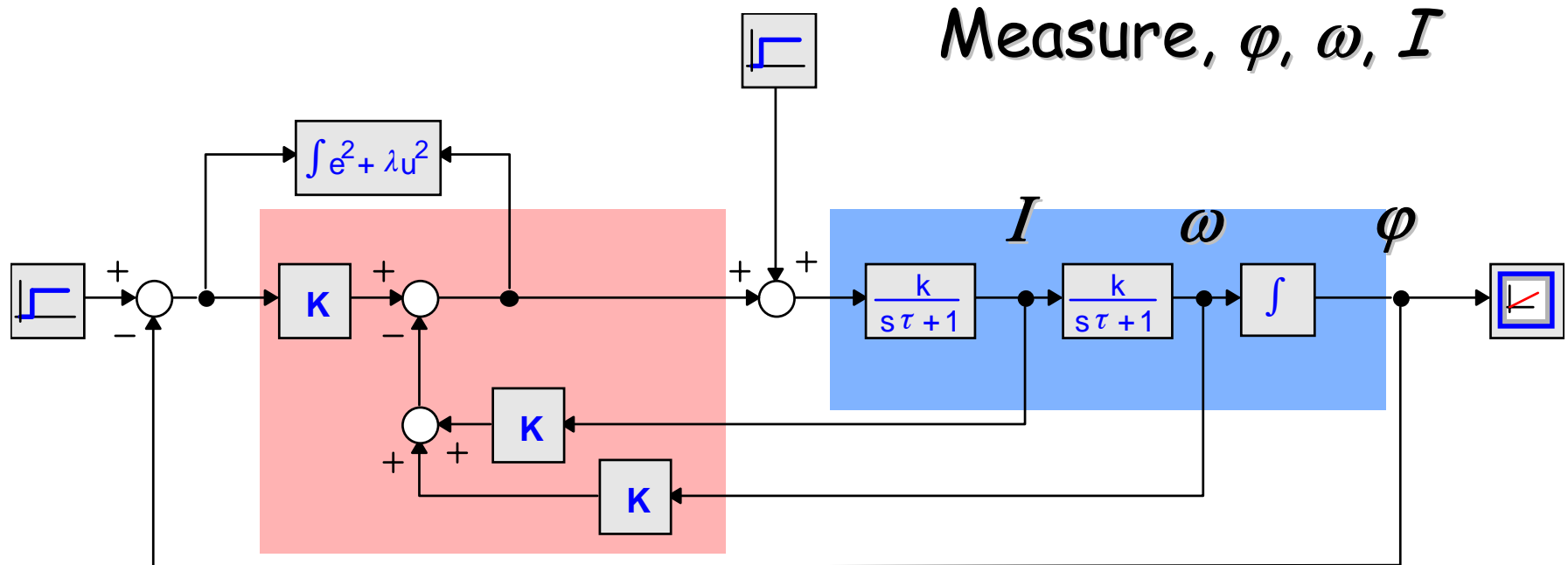
Reference $\neq 0$



Reference $\neq 0$

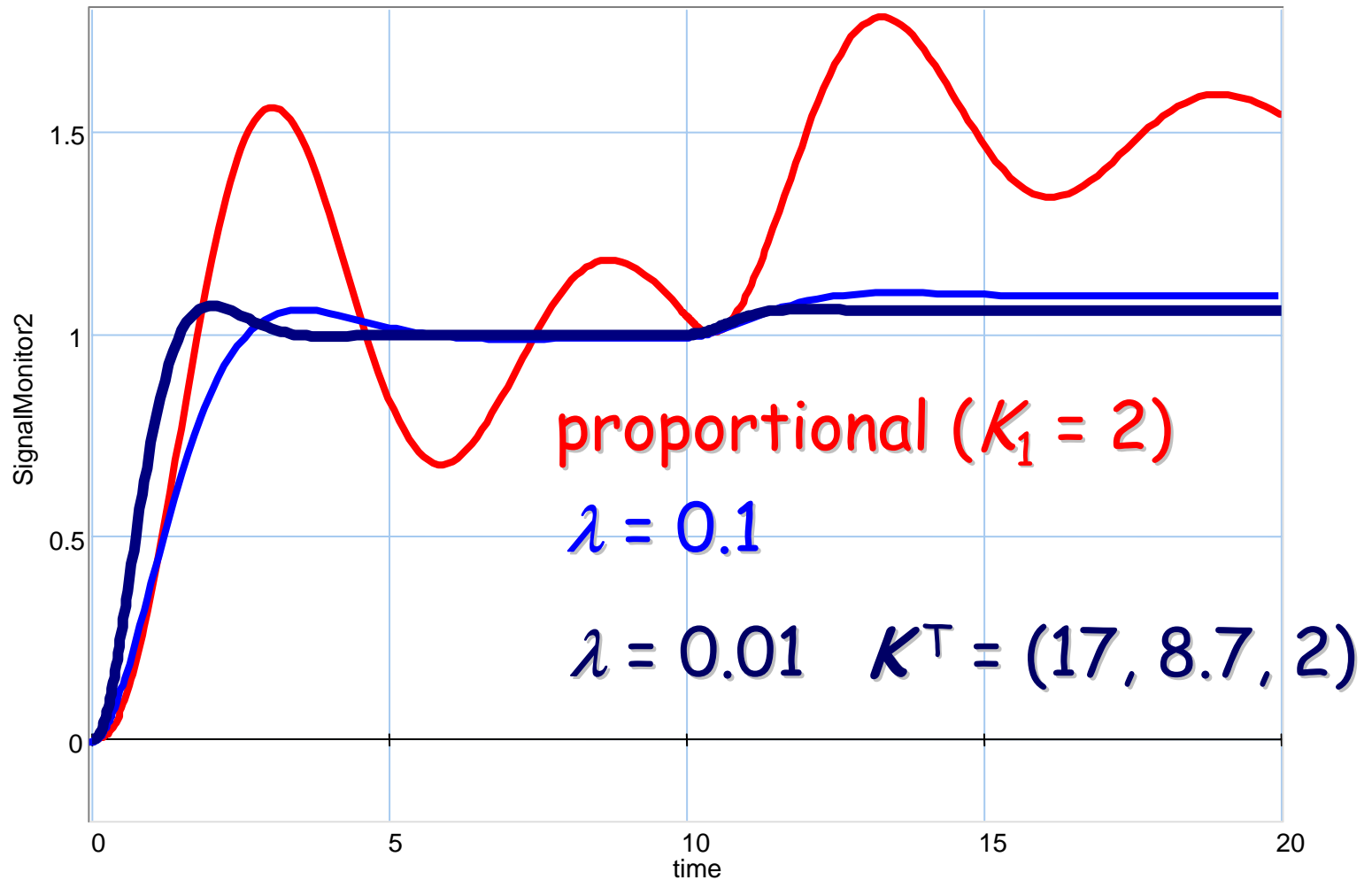


Example robot link



Exercise: do this optimisation yourself
(process parameters as in
sheet 3 of s-plane design)

20-sim



- State feedback
 - allows **poles** to be placed at any desired location
 - specially suited for computer-supported design
 - requires that all states be available
 - this is not always the case
 - may require state estimation